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Conrad

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(54) **CYCLONIC AIR TREATMENT MEMBER AND SURFACE CLEANING APPARATUS INCLUDING THE SAME**

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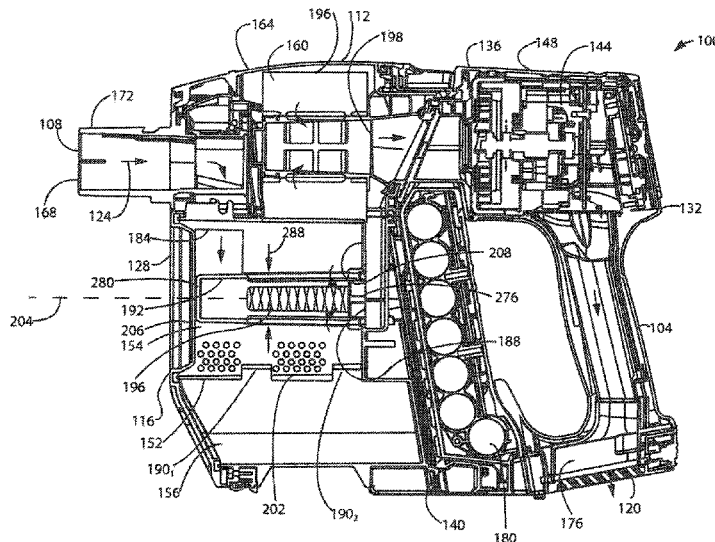
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(57) **ABSTRACT**

A cyclonic air treatment member comprises a cyclone and a dirt collection chamber external to the cyclone chamber. The dirt collection chamber has first and second discrete dirt outlet regions, each dirt outlet region extending around a portion of the perimeter of the cyclone chamber. The second dirt outlet region is positioned proximate the cyclone second end, and the first dirt outlet region is positioned toward the cyclone first end relative to the second dirt outlet region.

25 Claims, 27 Drawing Sheets



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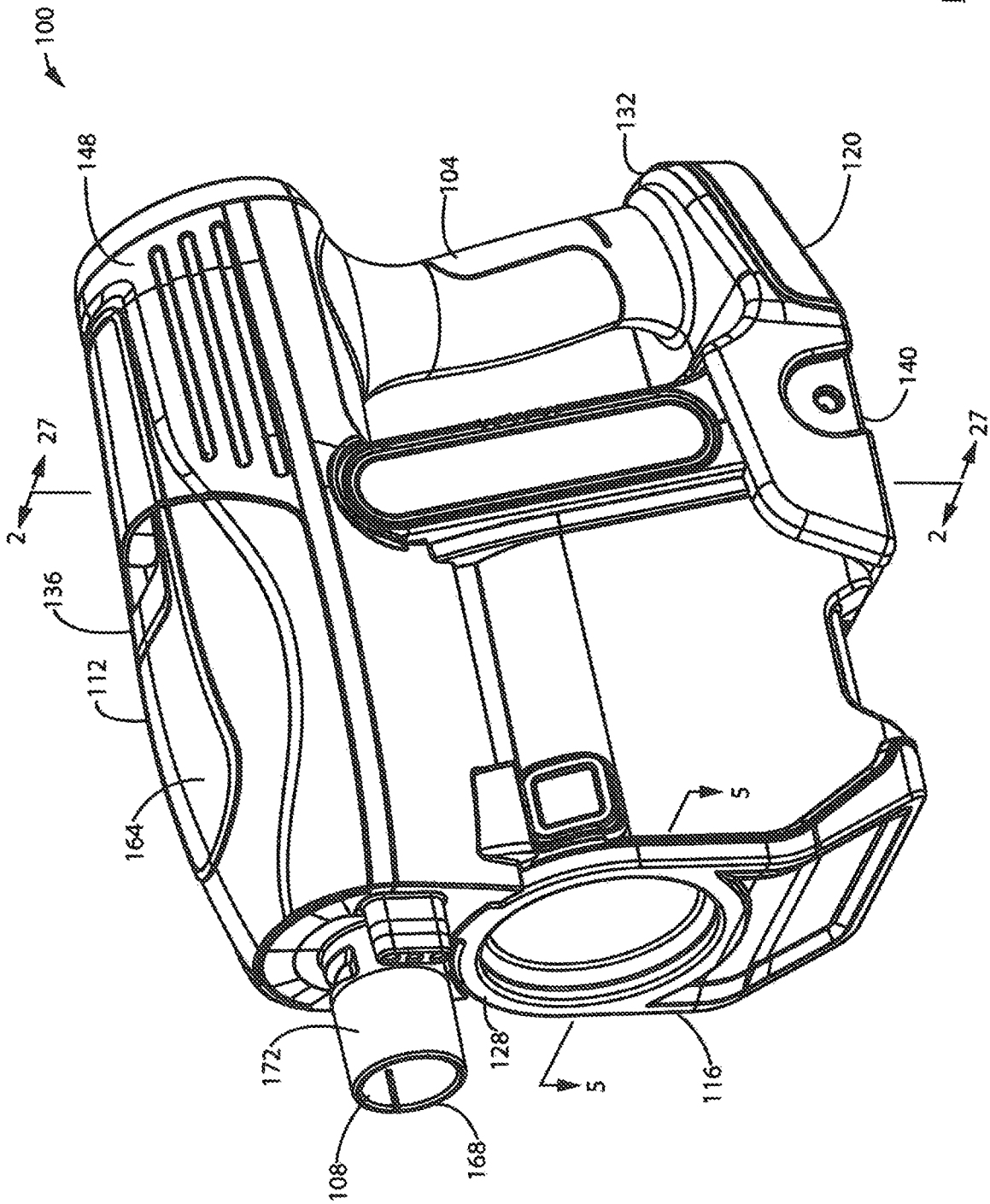


FIG. 1

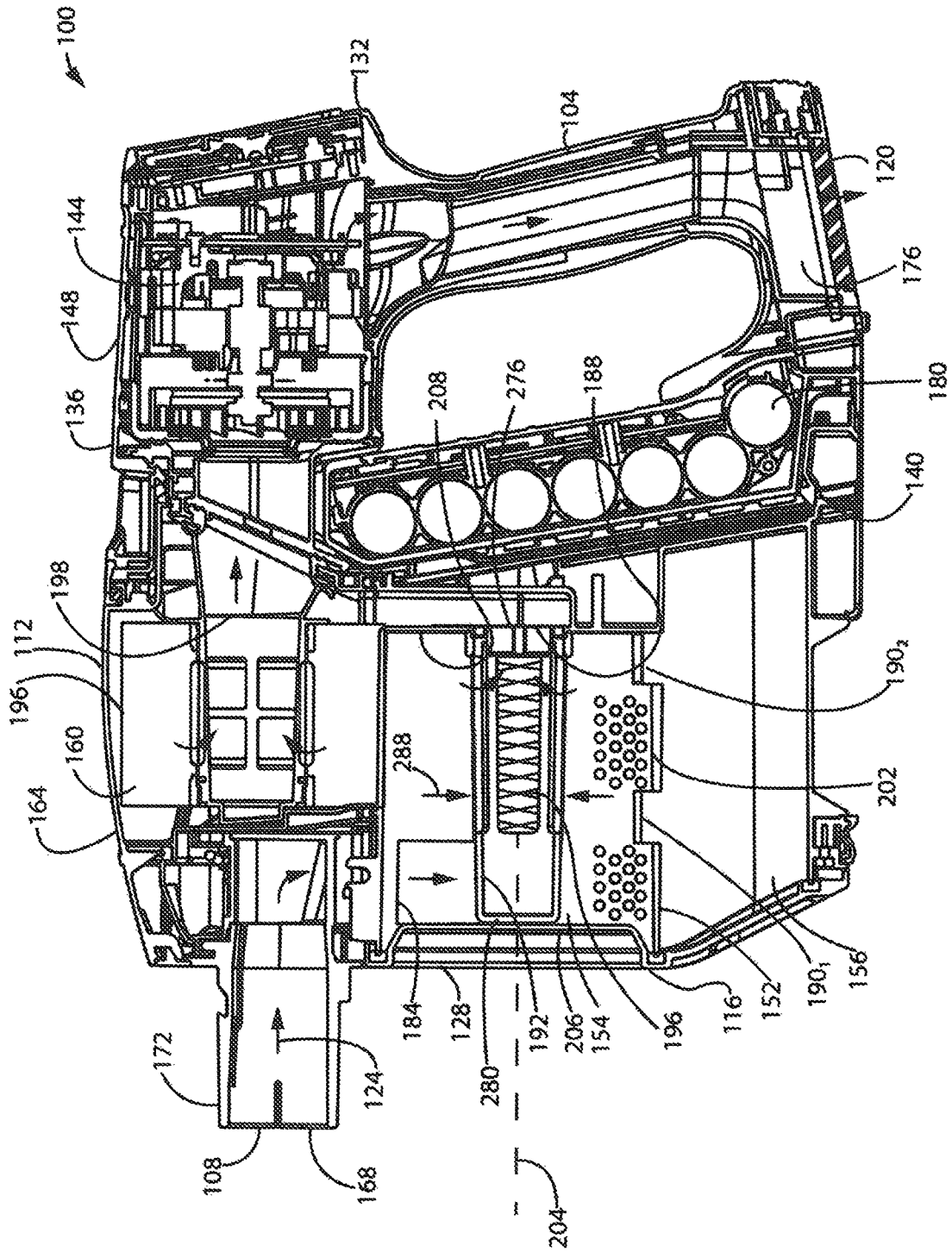


FIG. 2

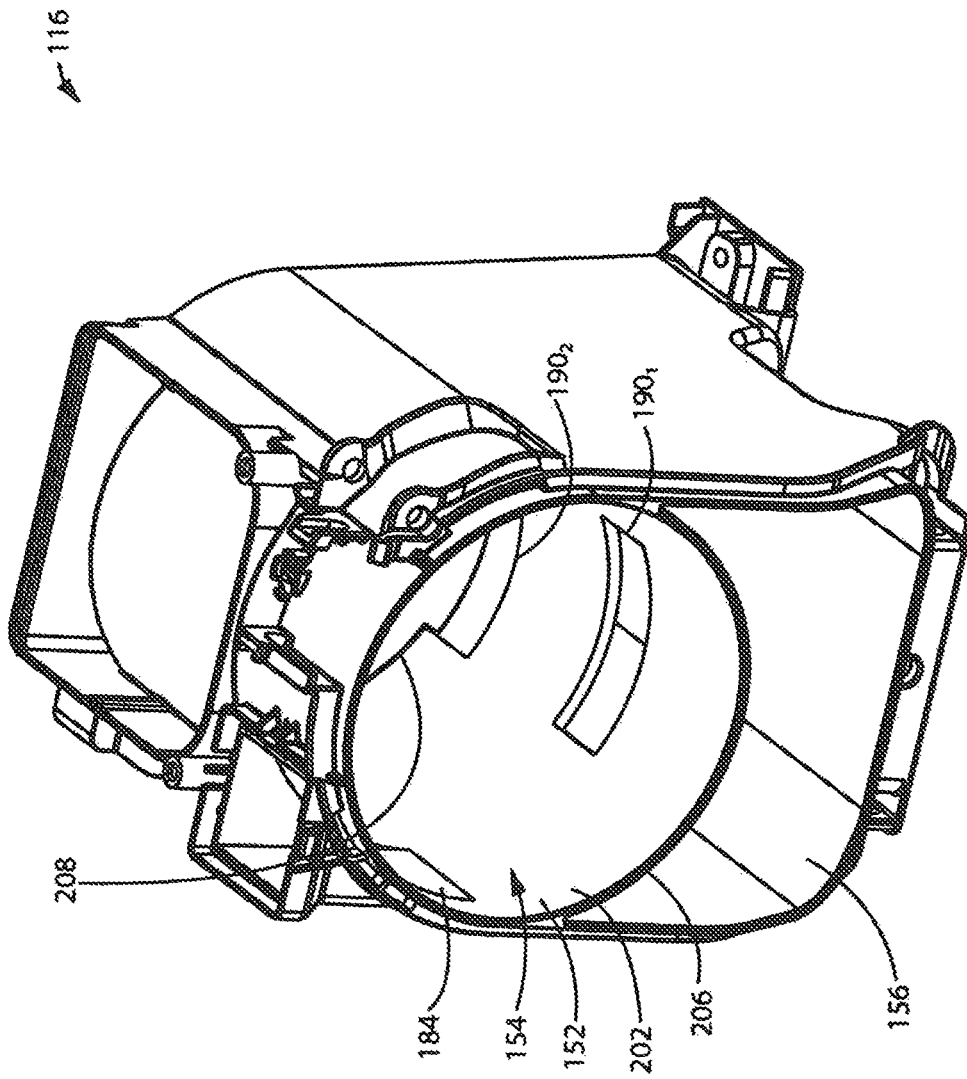


FIG. 3

116

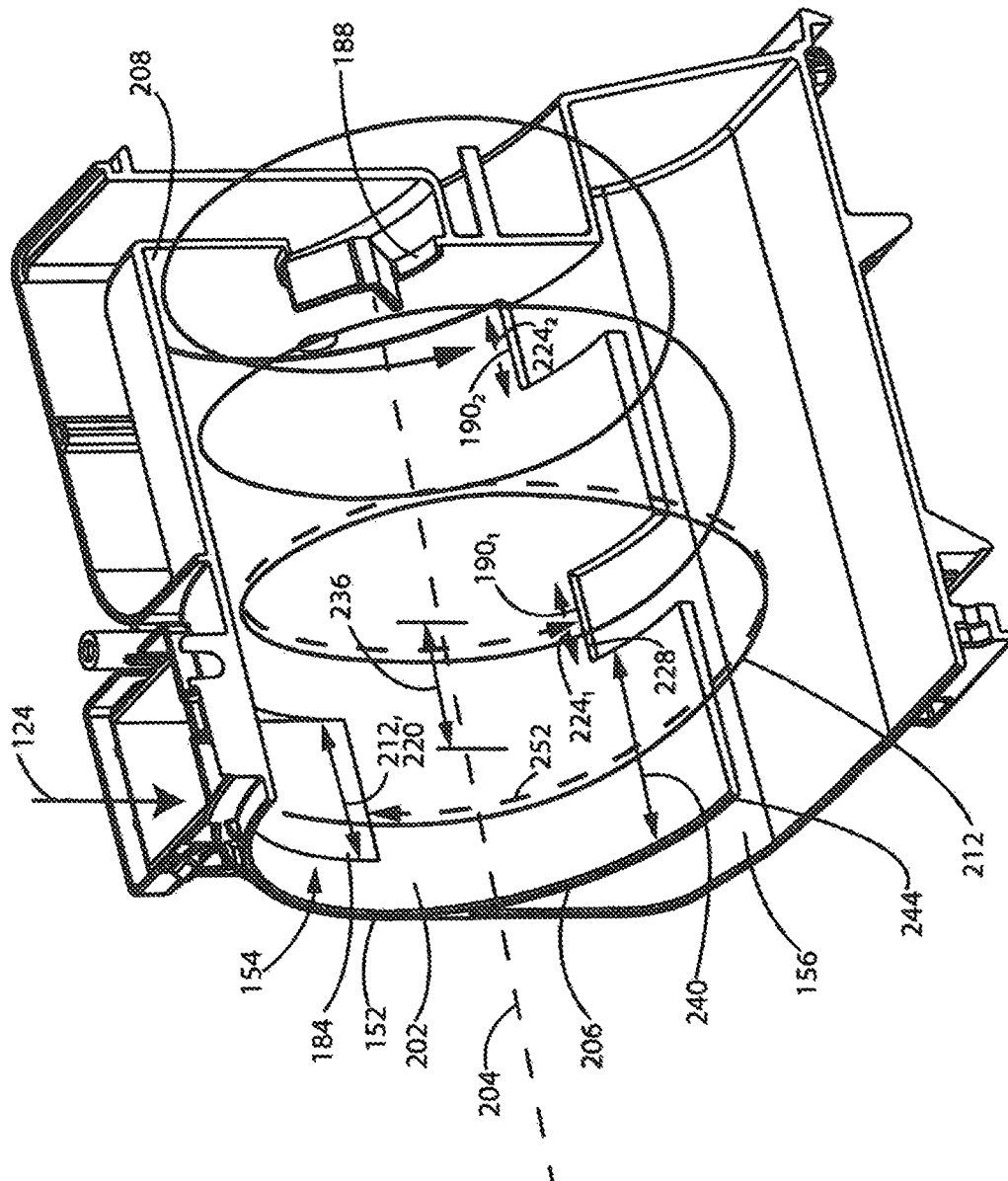


FIG. 4

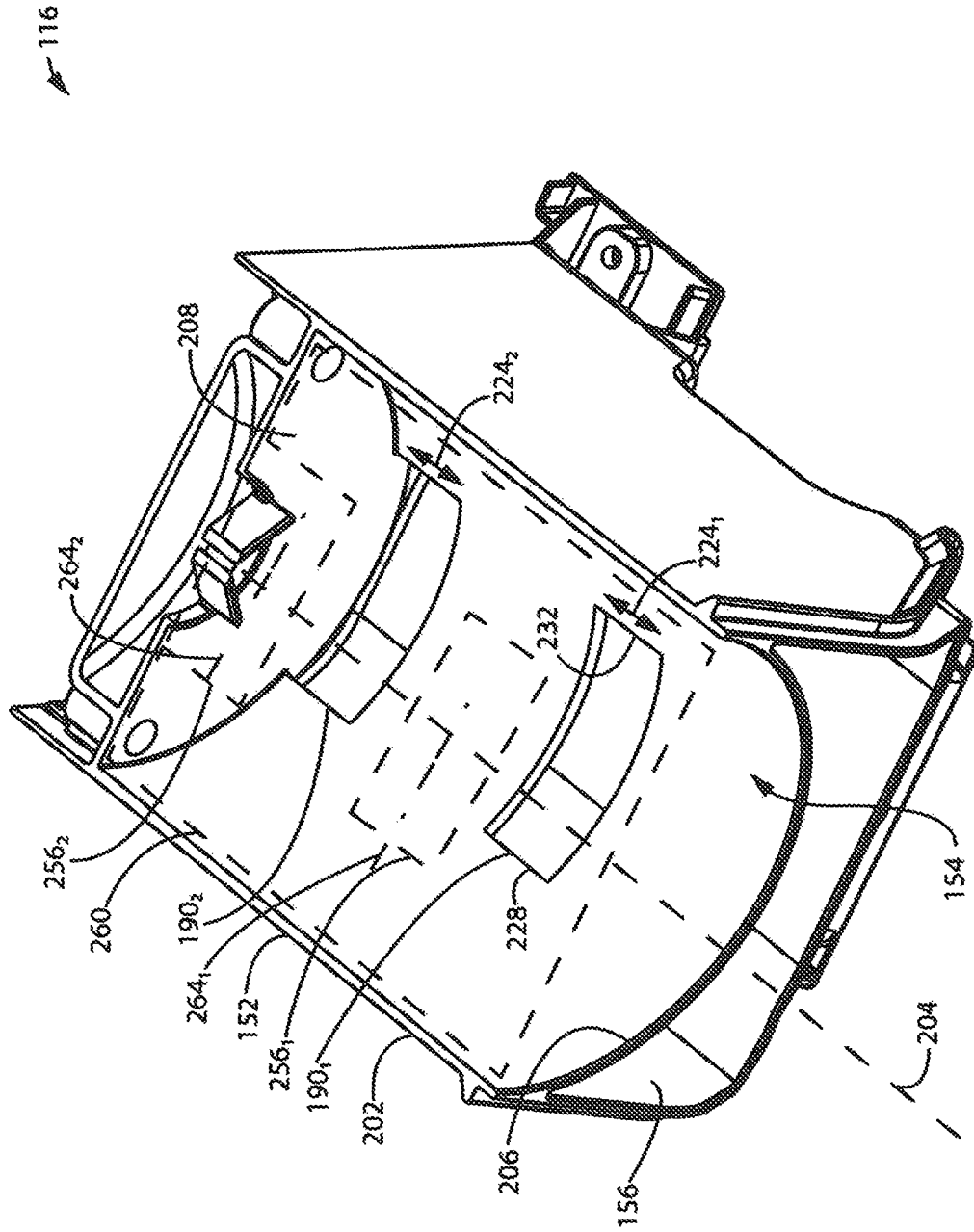


FIG. 5

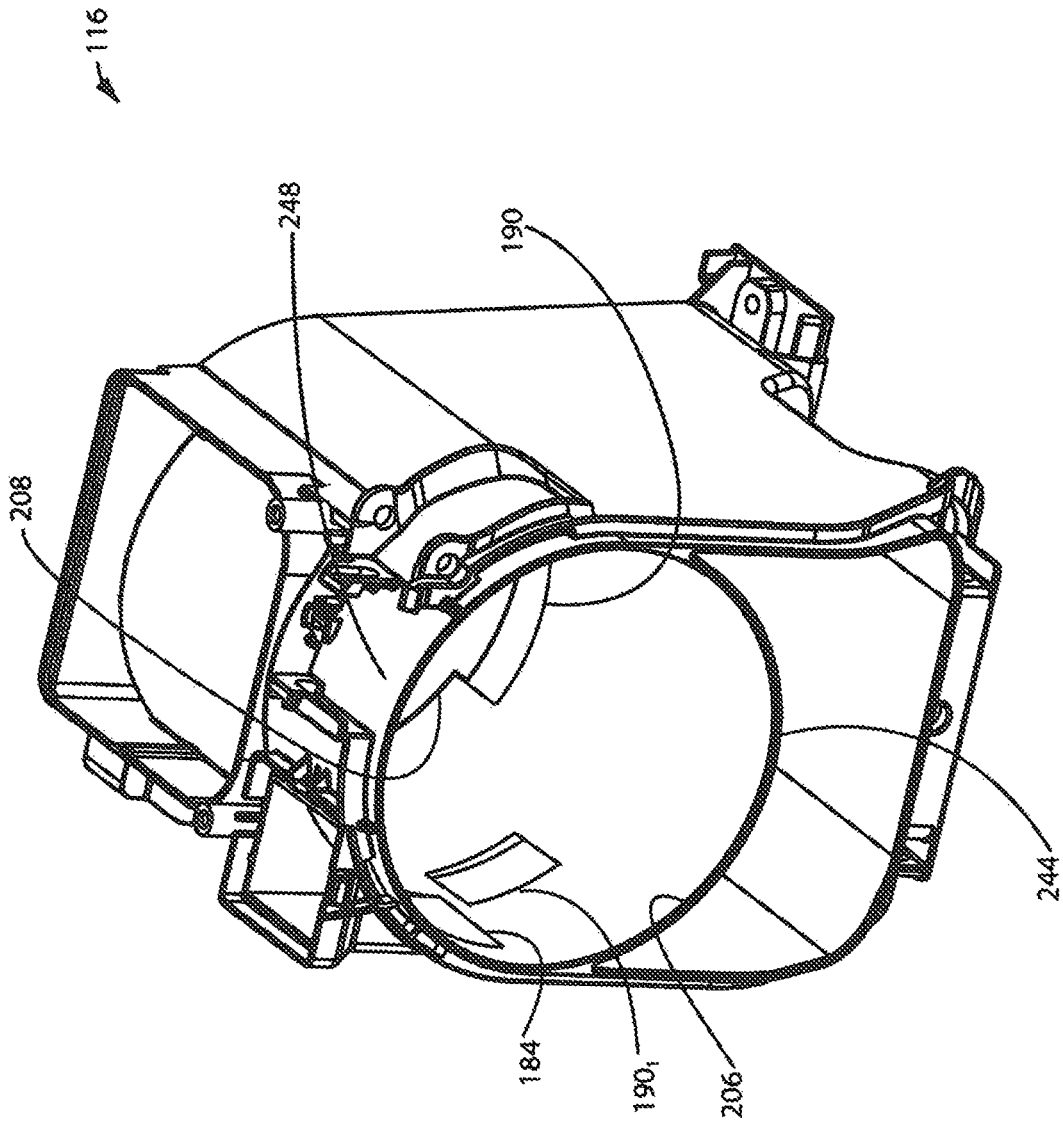


FIG. 6

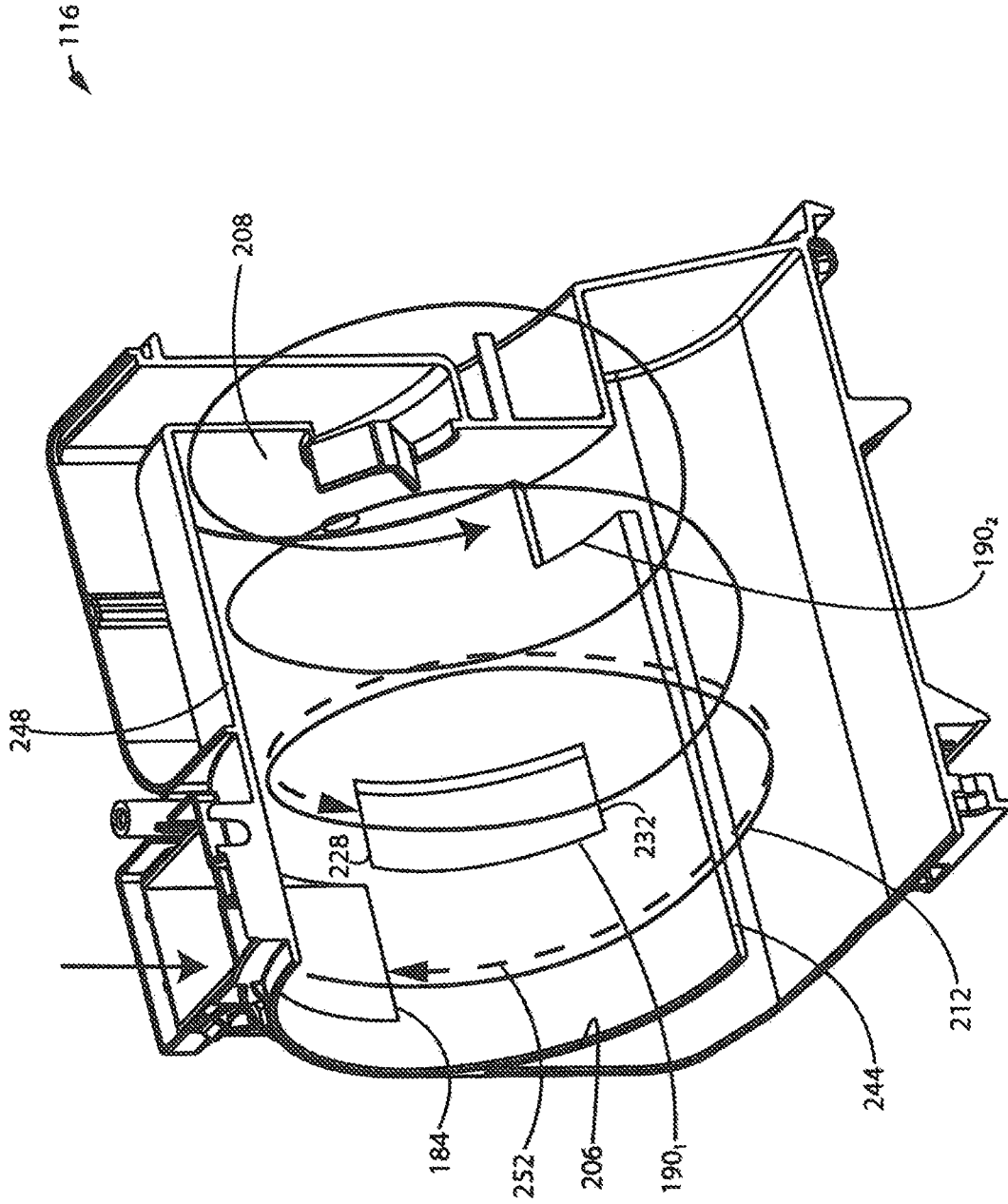


FIG. 7

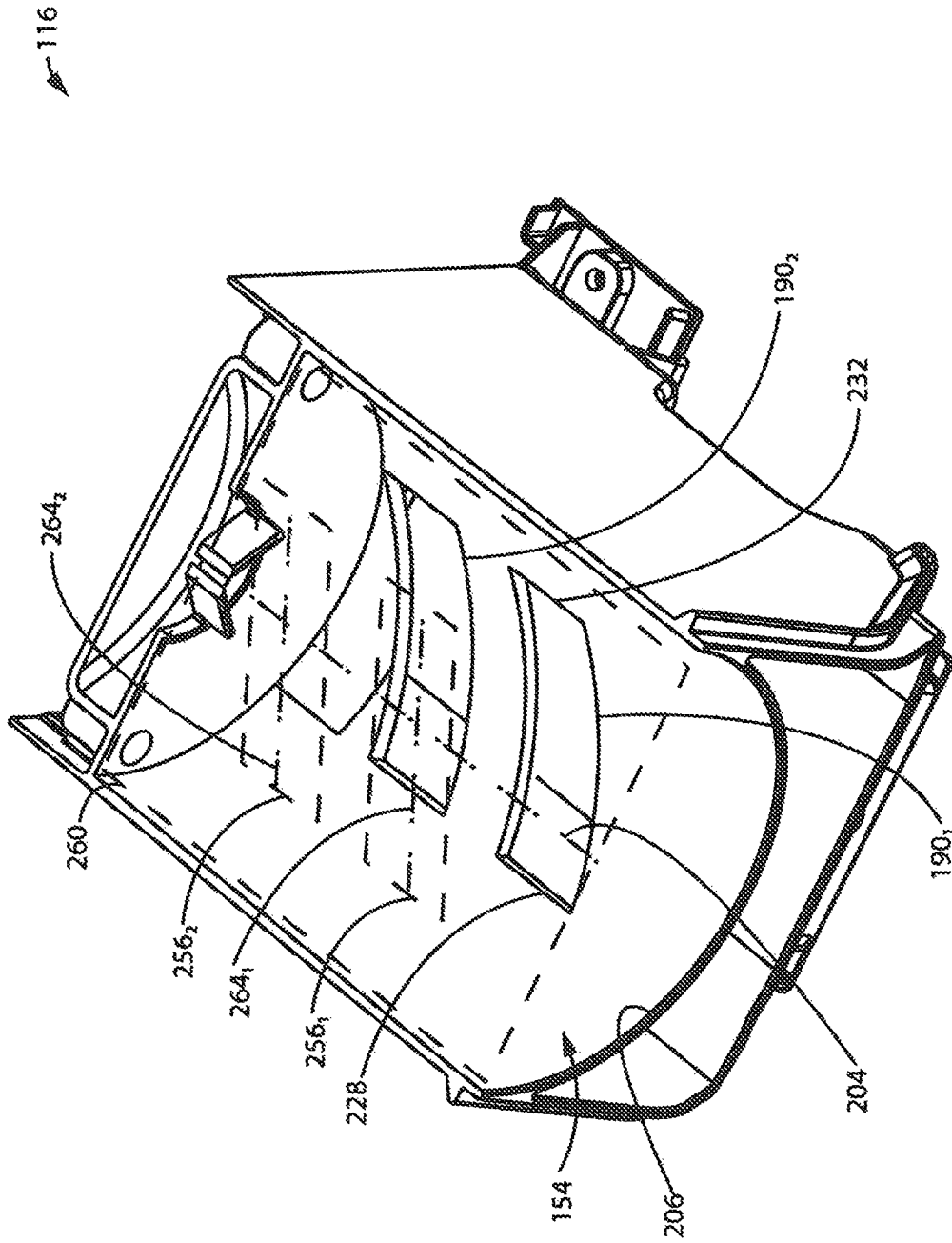


FIG. 8

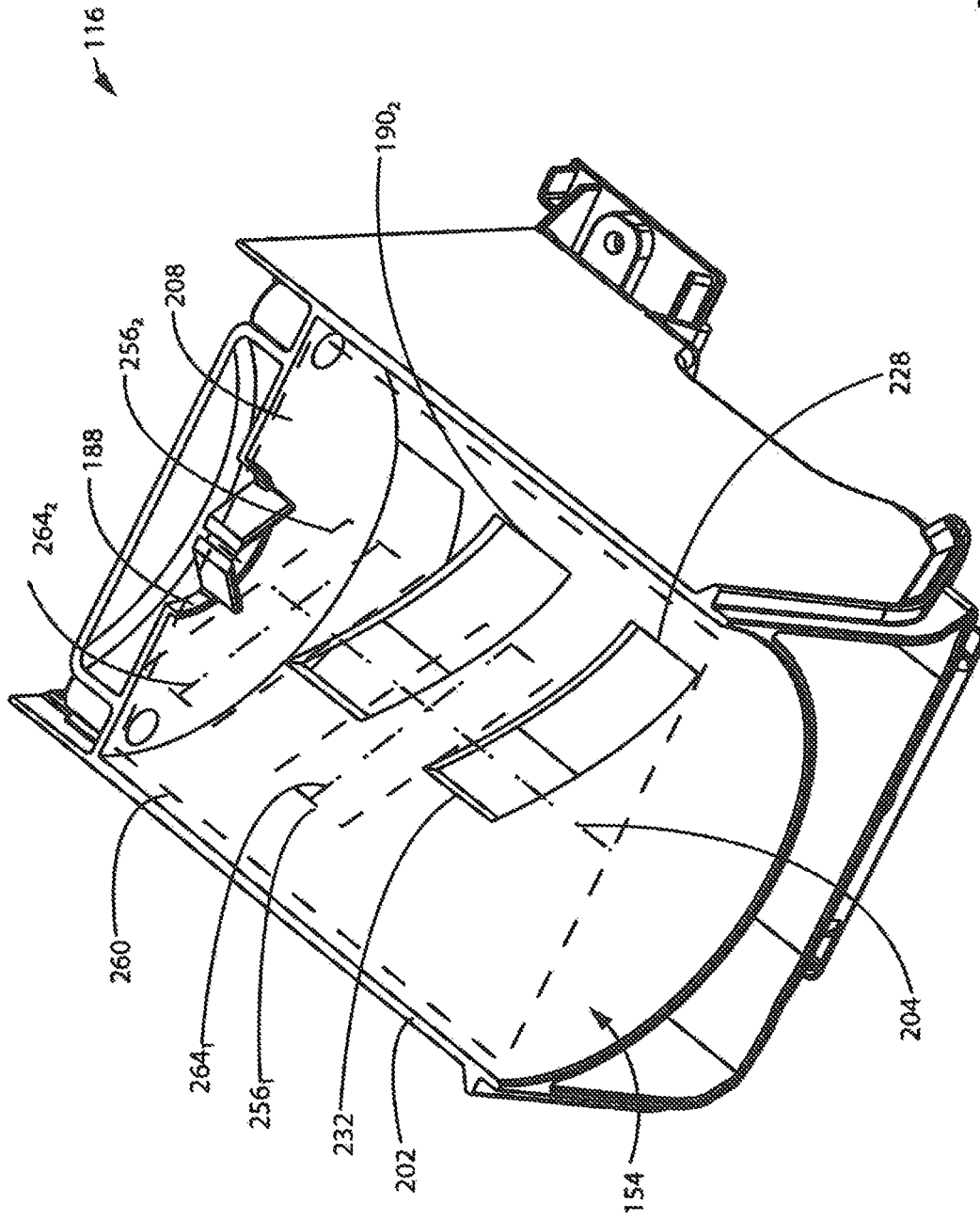


FIG. 9

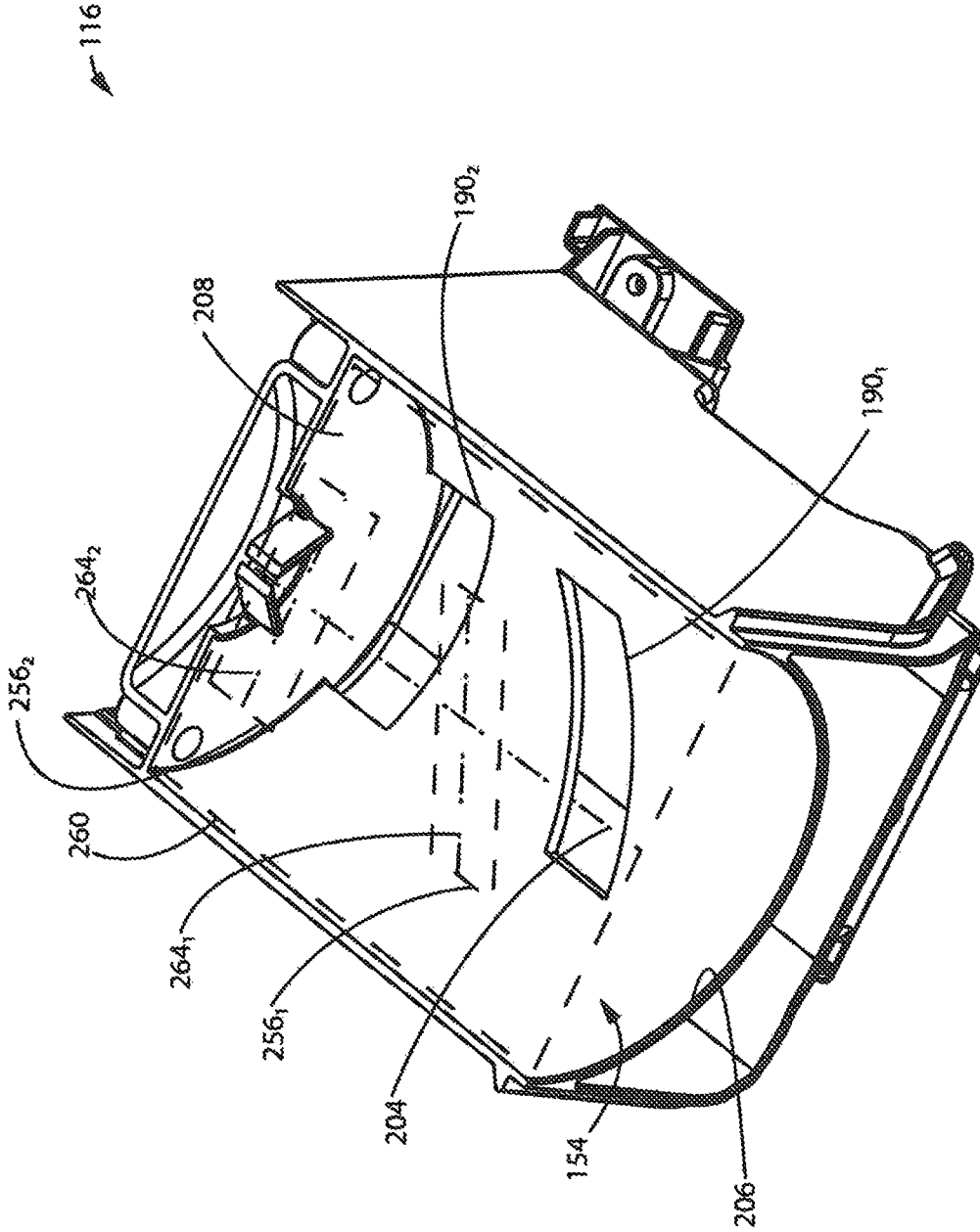


FIG. 10

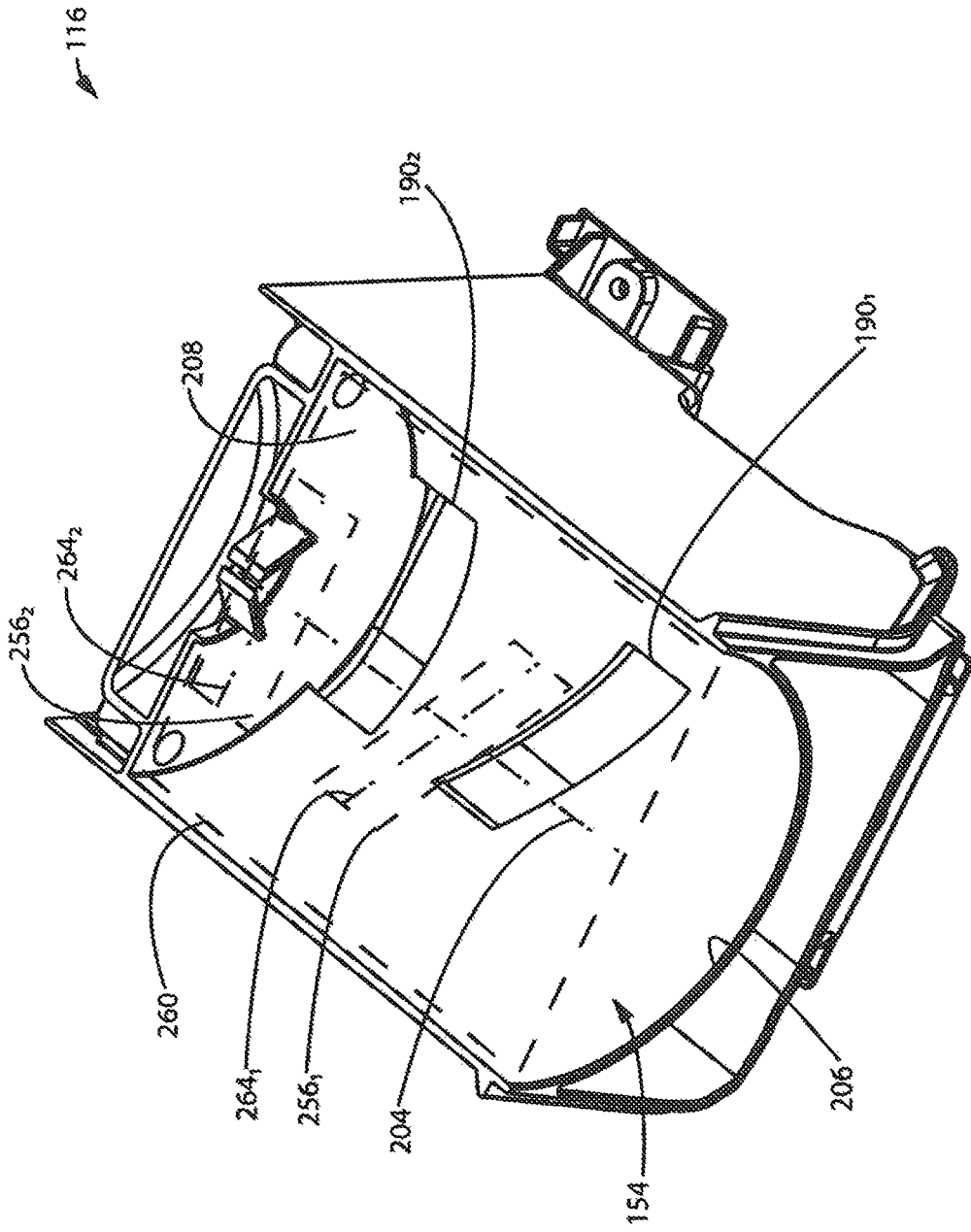


FIG. 11

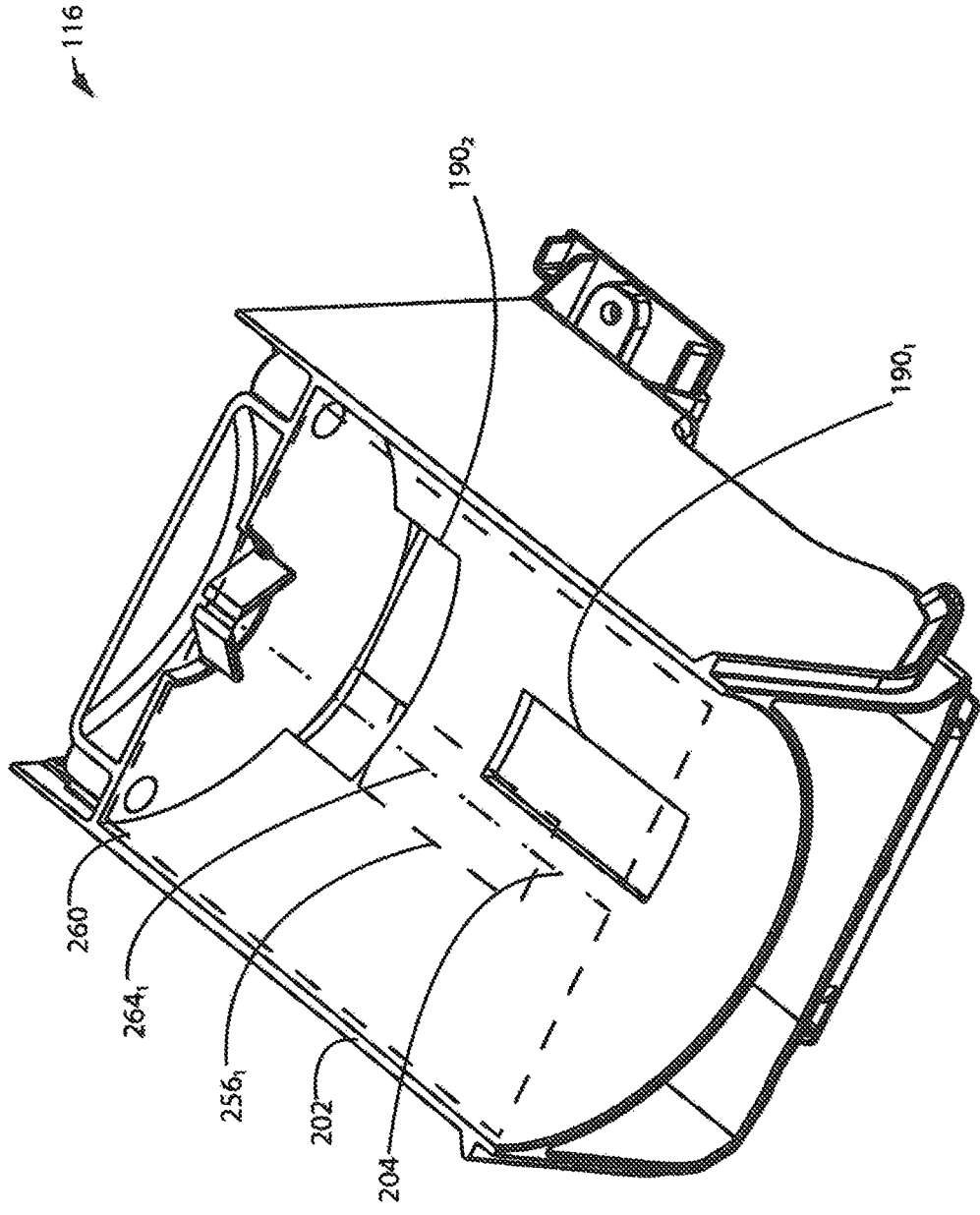


FIG. 12

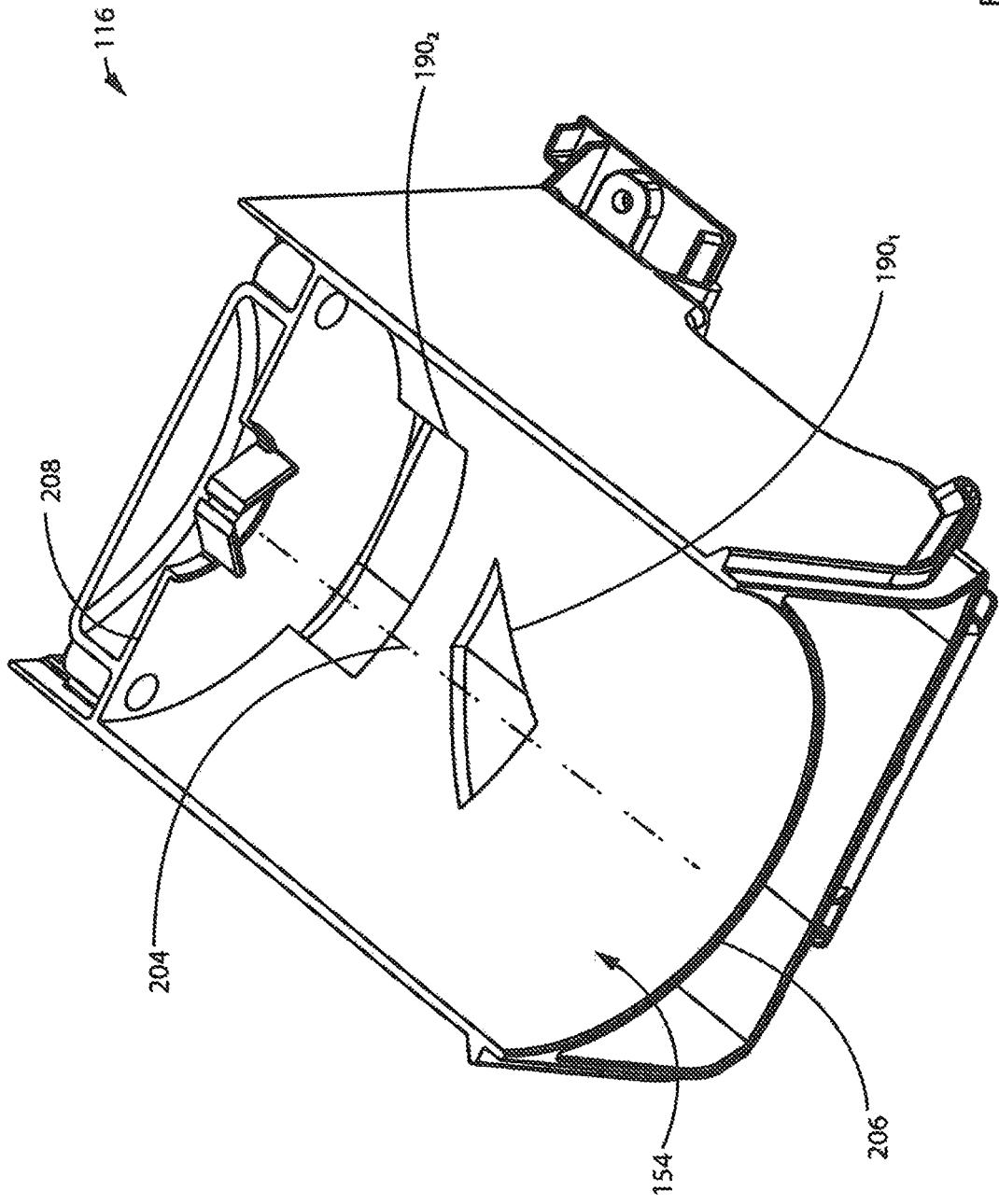


FIG. 13

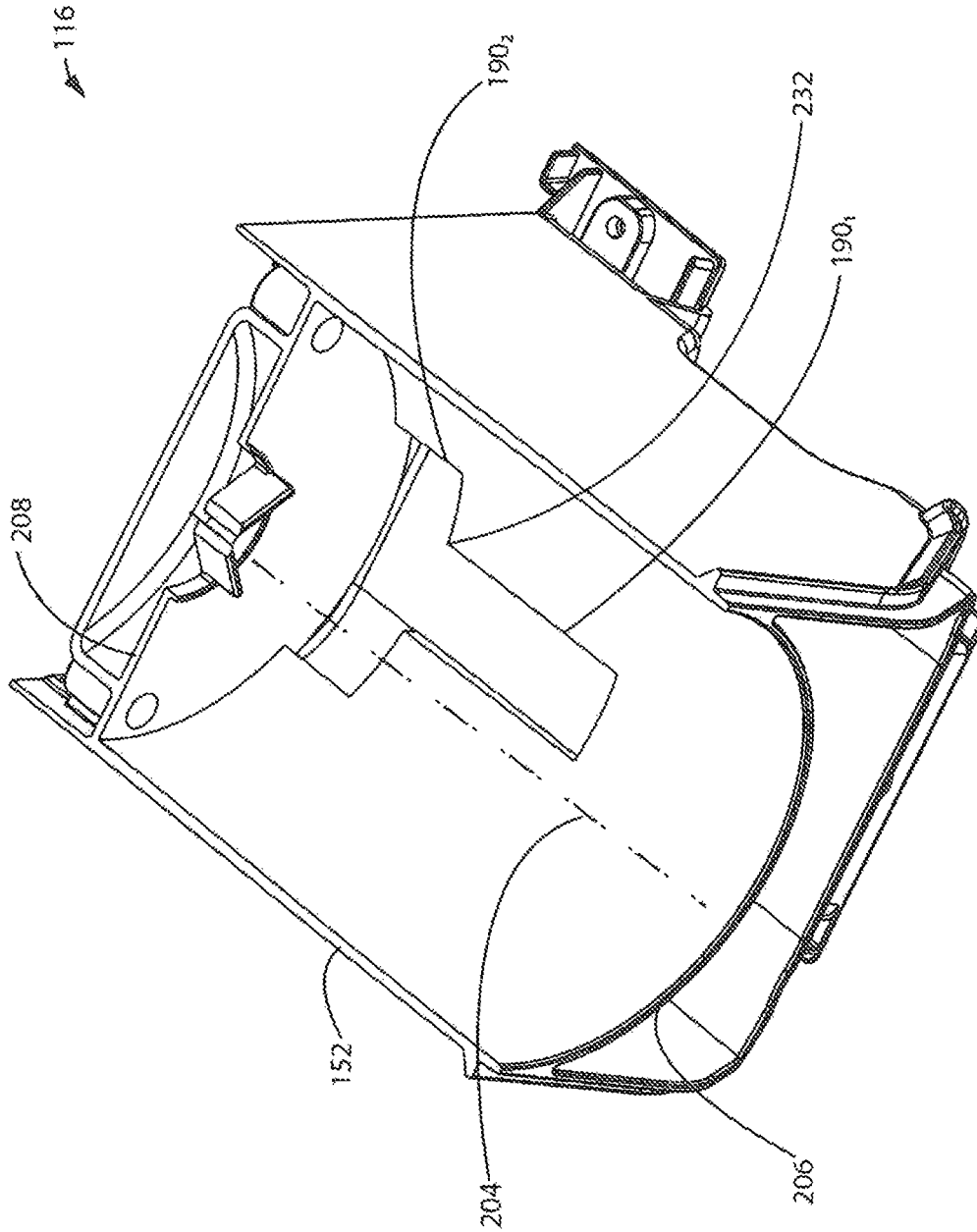


FIG. 14

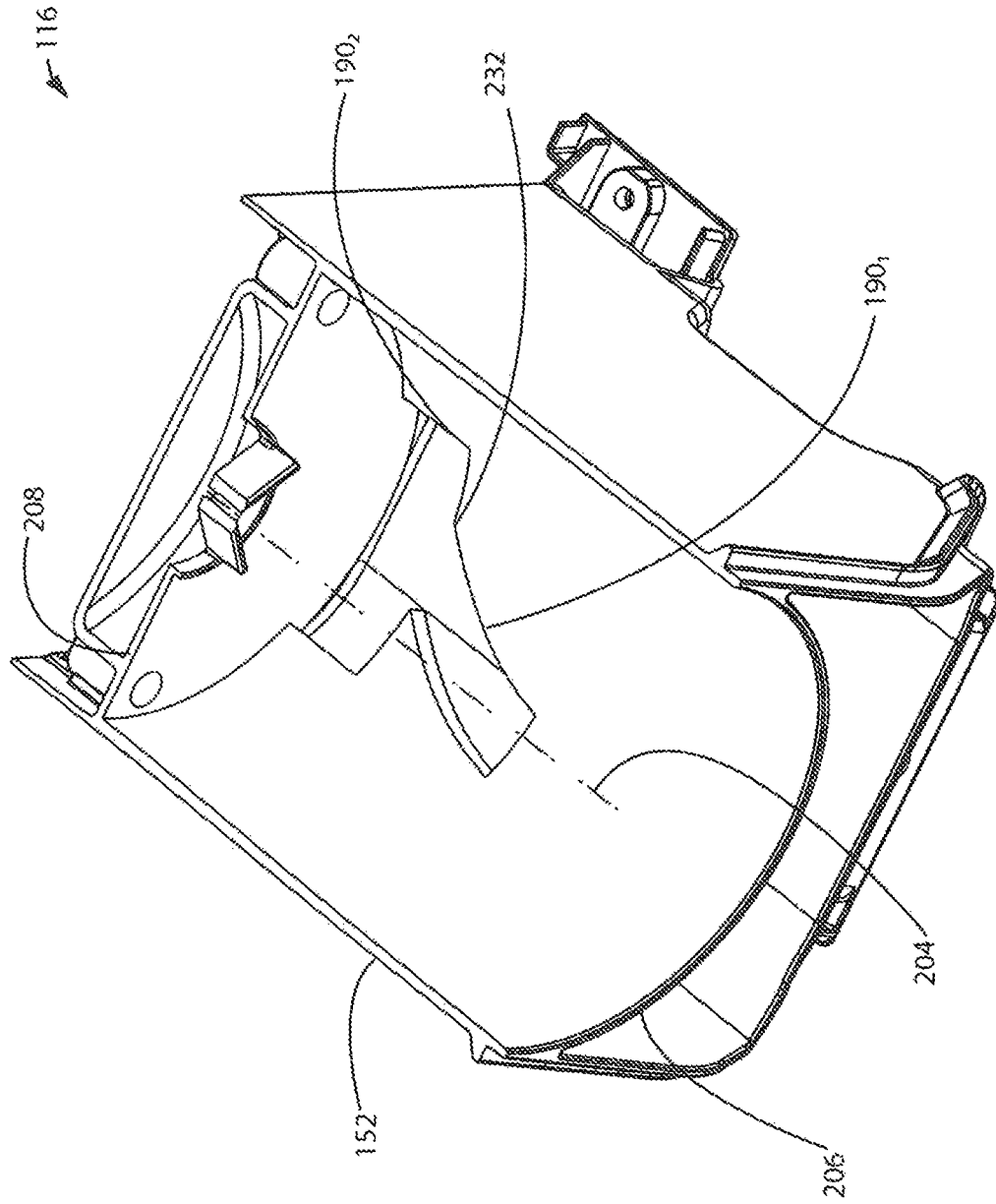


FIG. 15

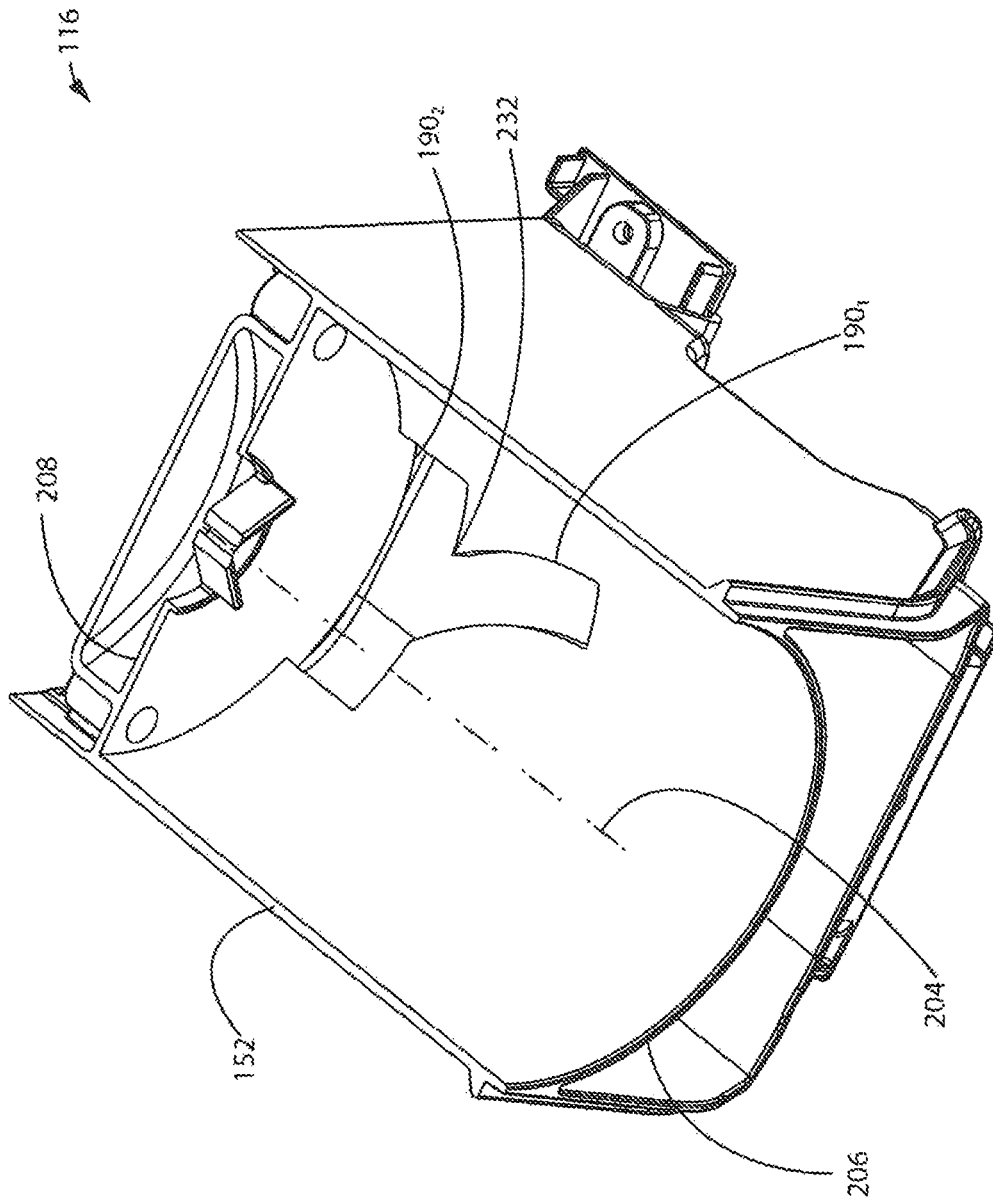


FIG. 16

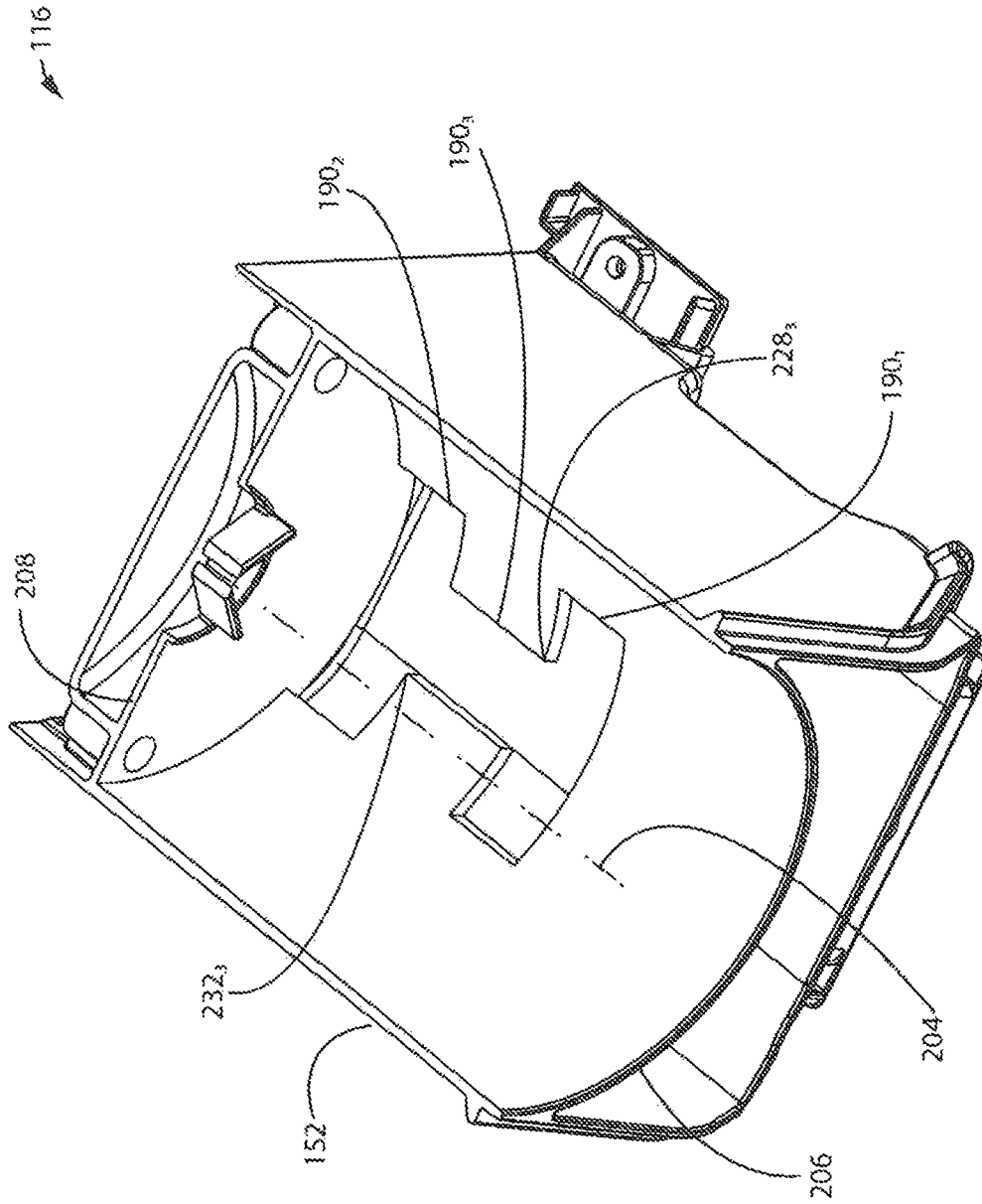


FIG. 17

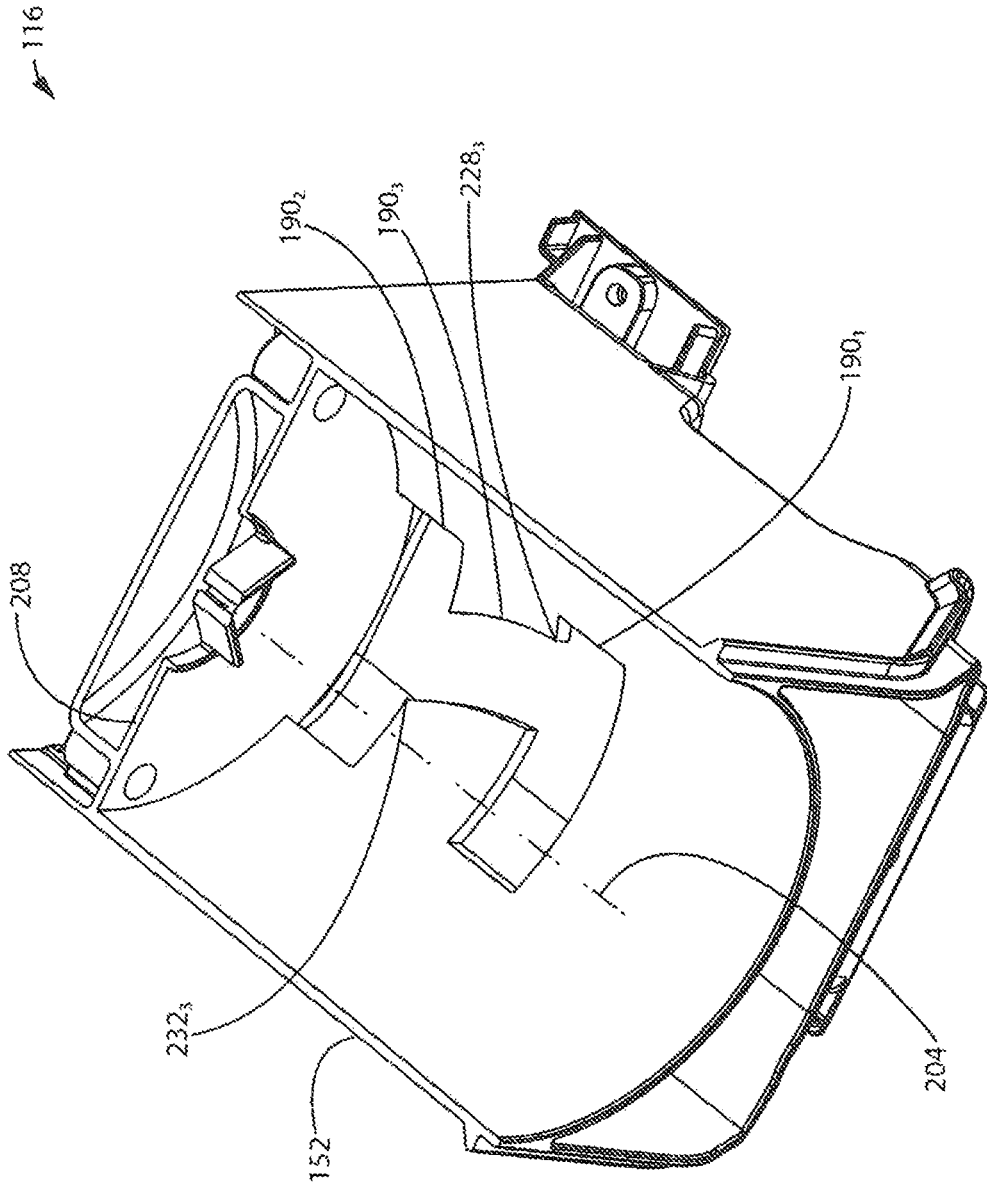


FIG. 18

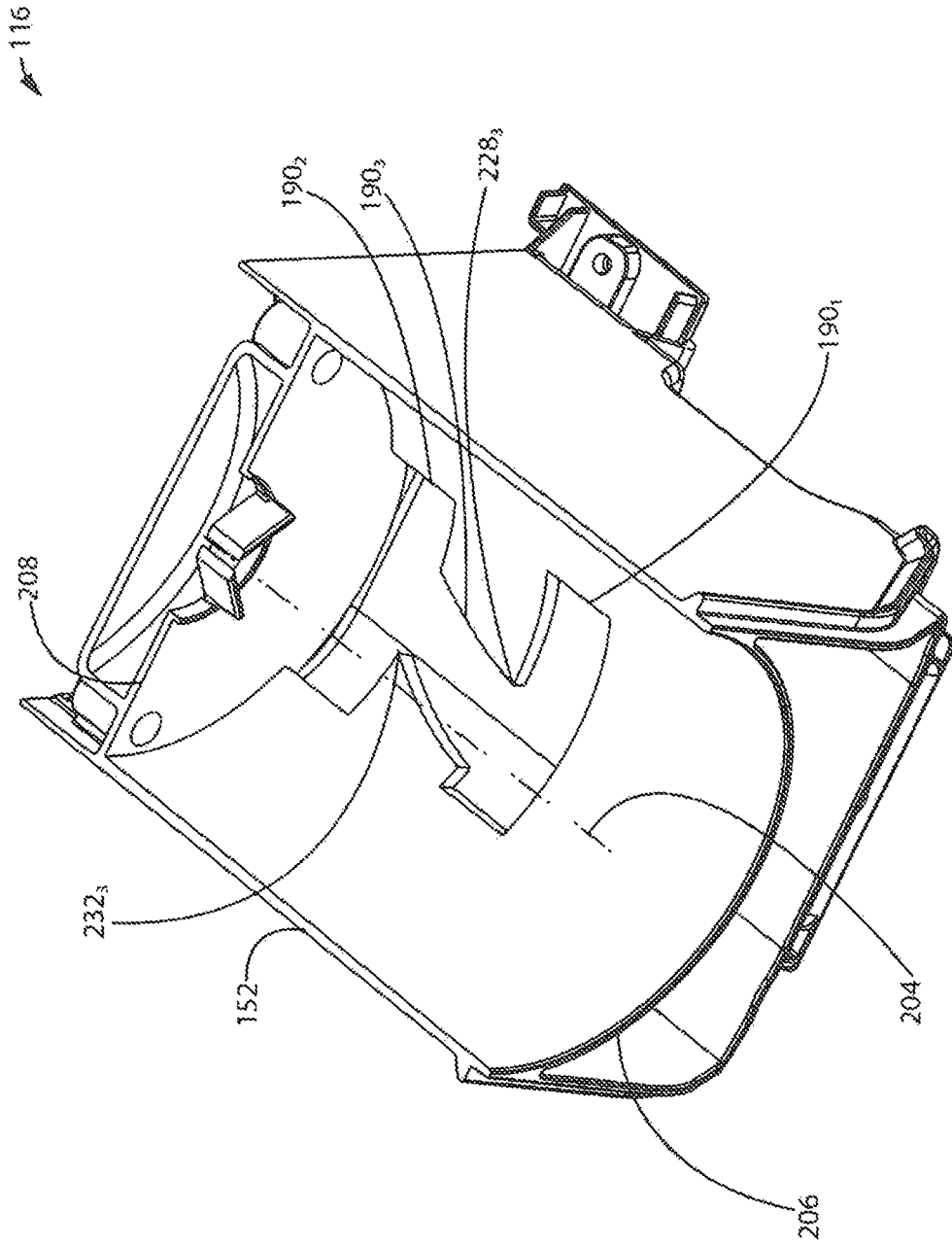


FIG. 19

116

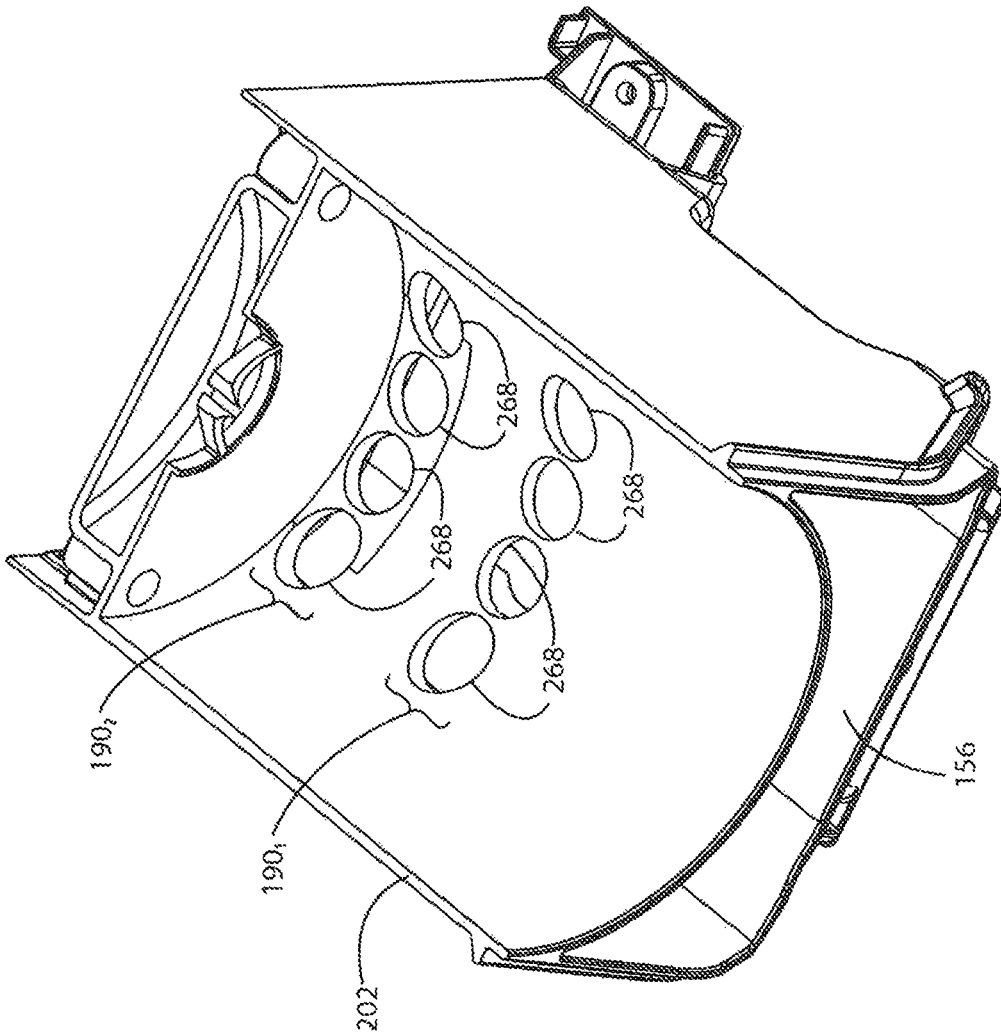


FIG. 20

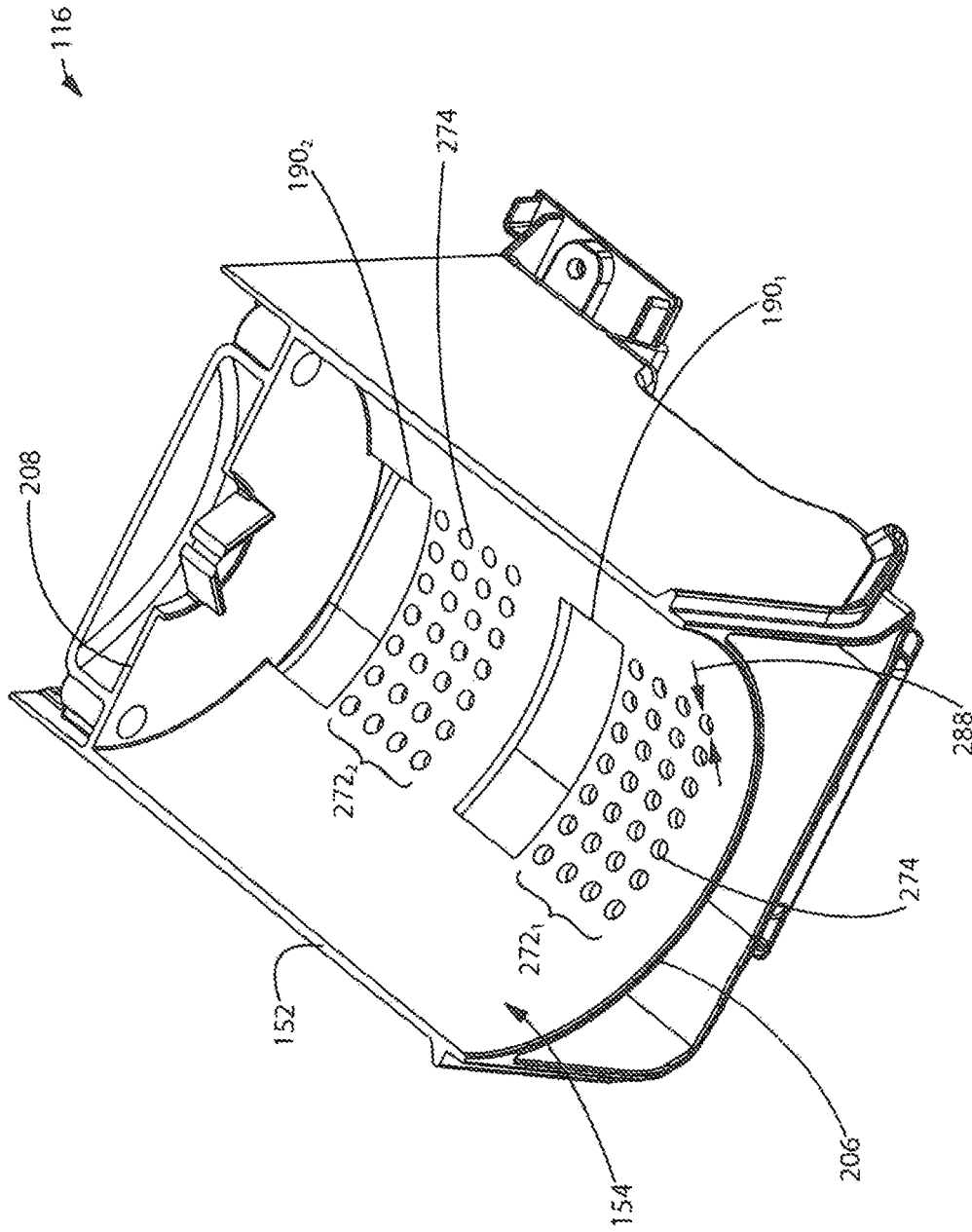


FIG. 21

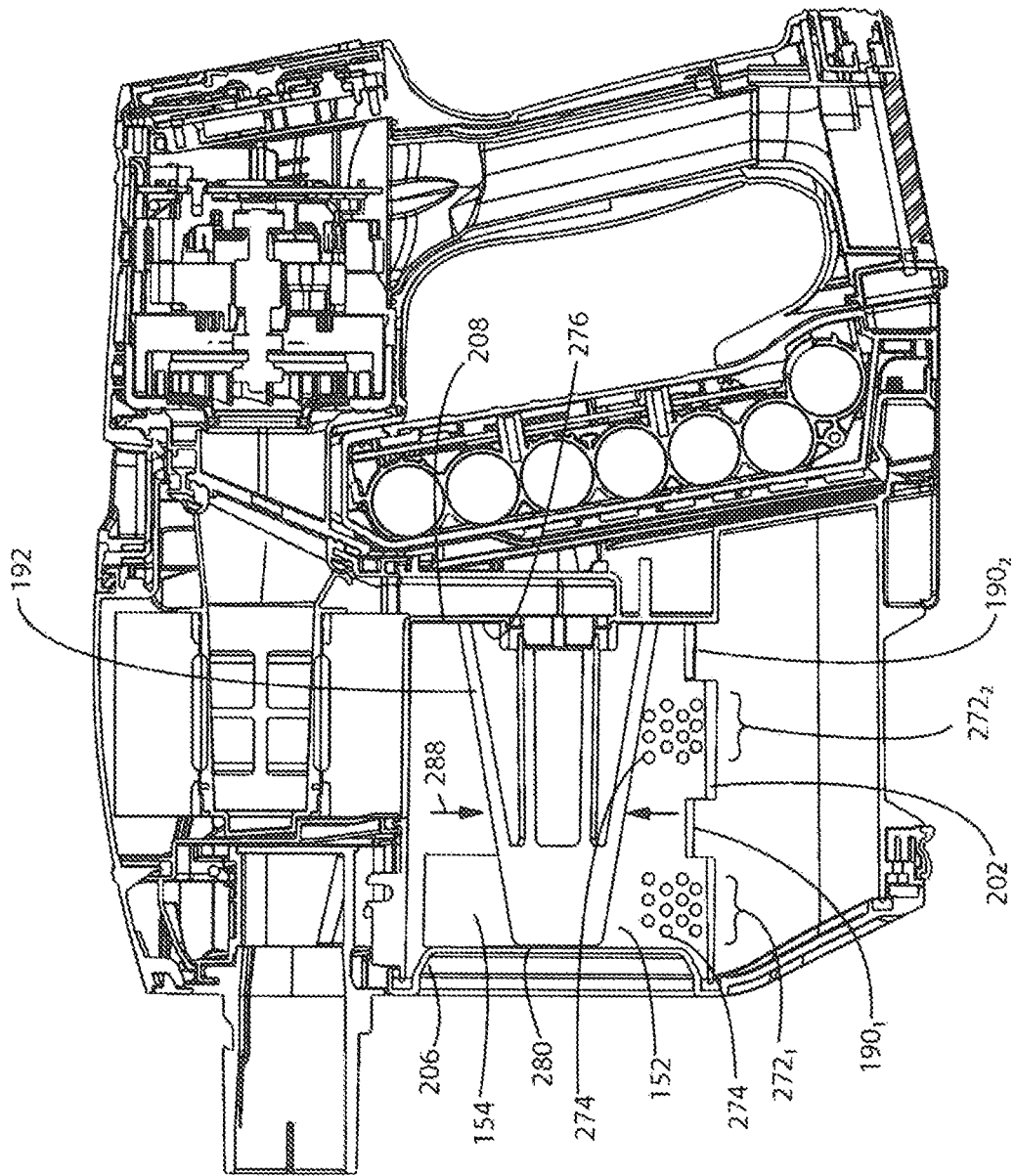


FIG. 22

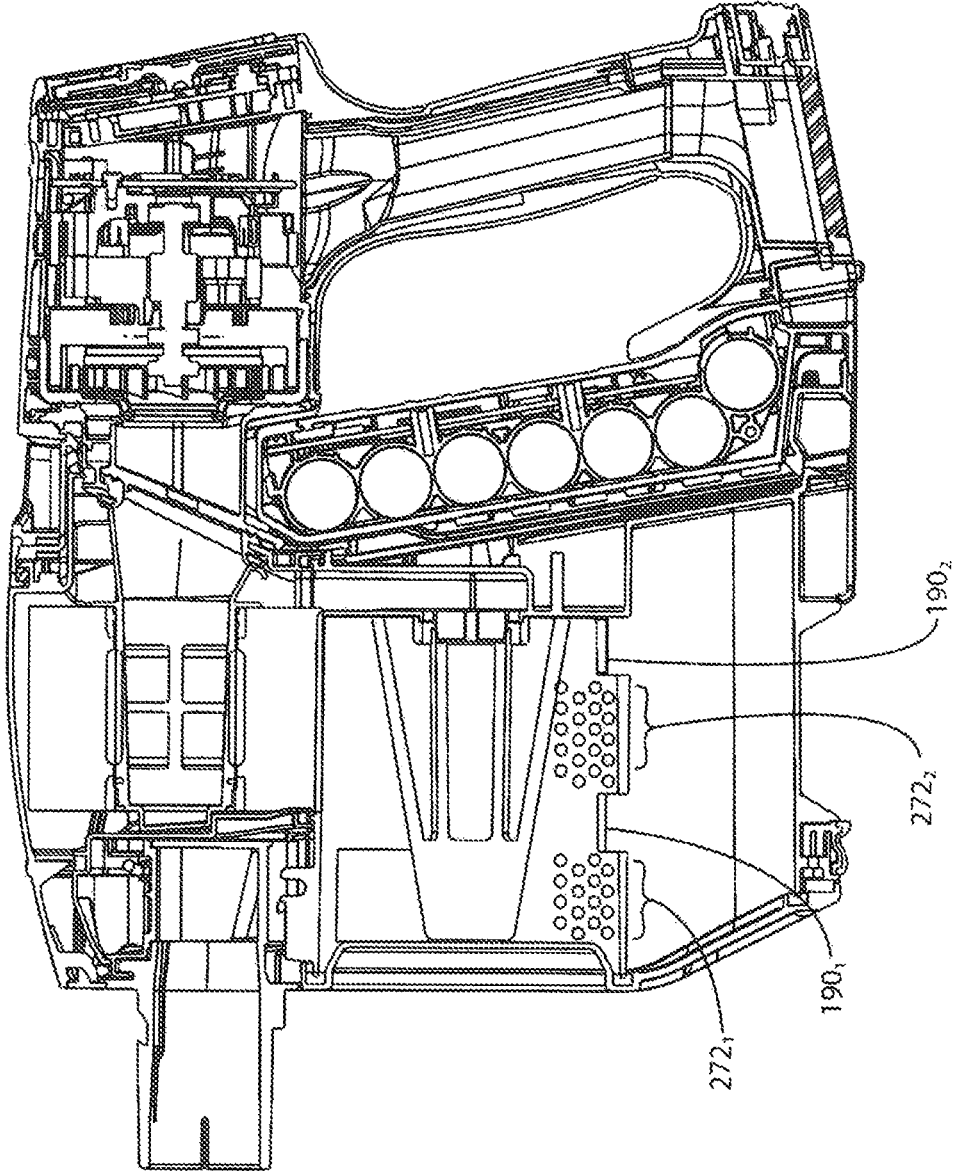


FIG. 23

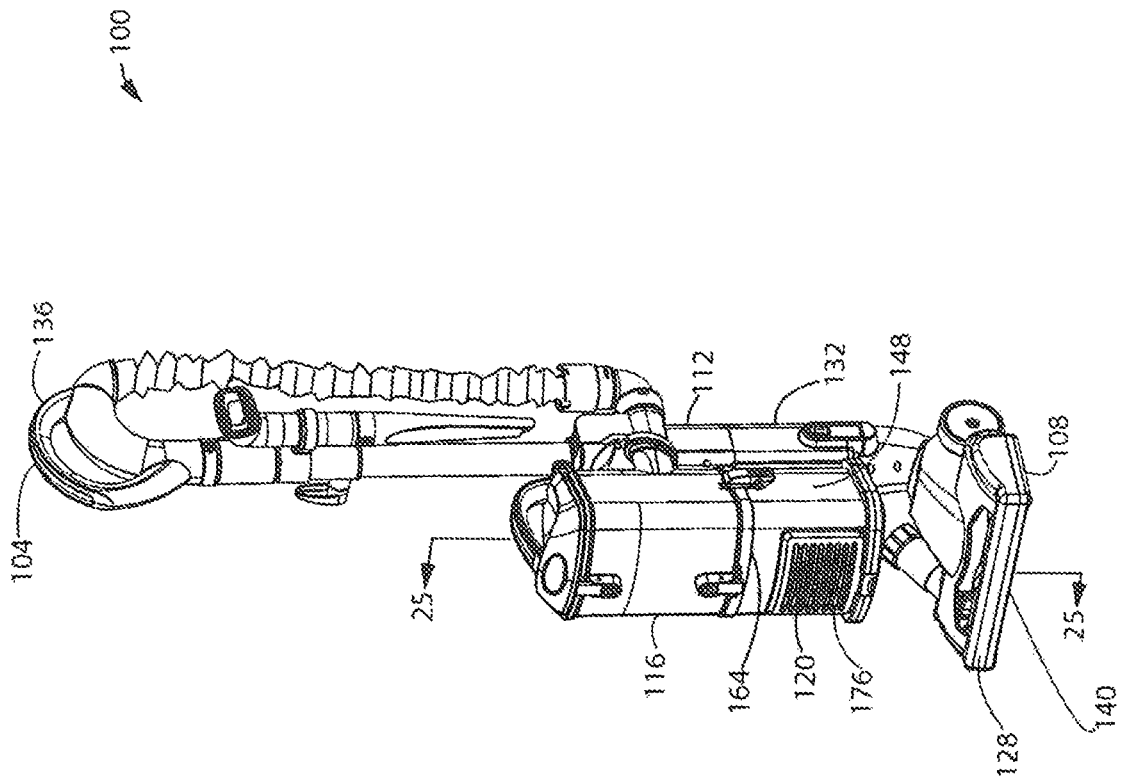


FIG. 24

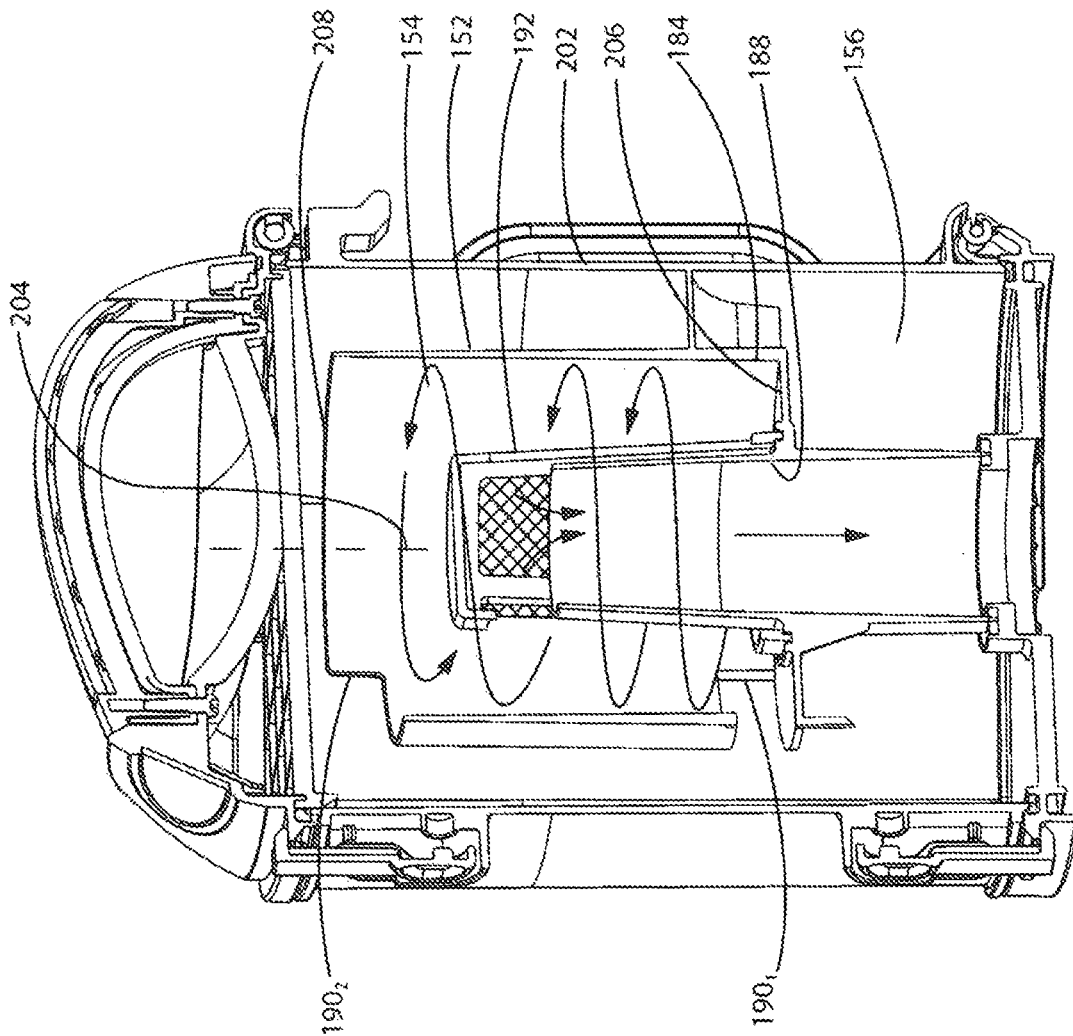
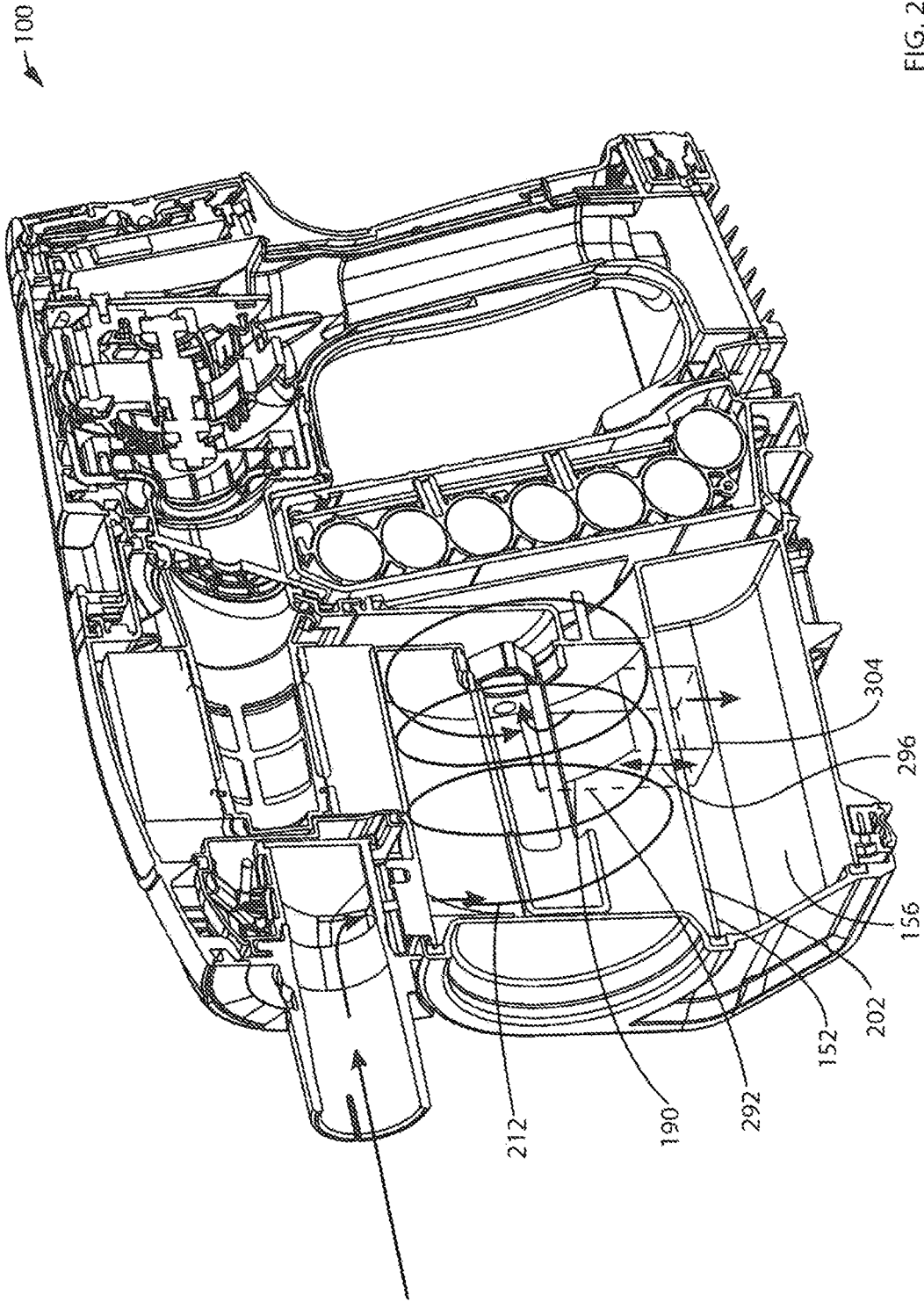


FIG. 25



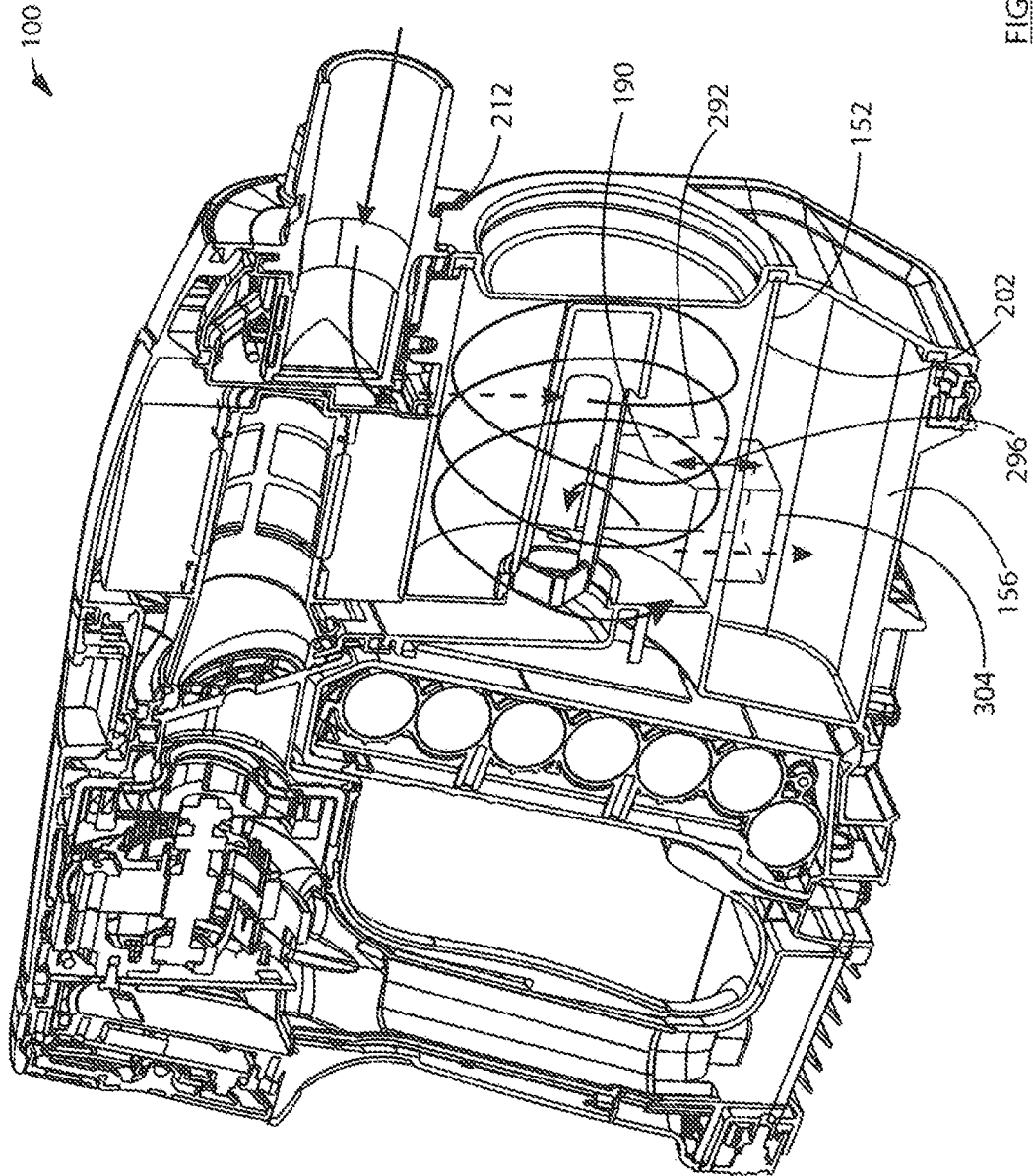


FIG. 27

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**CYCLONIC AIR TREATMENT MEMBER
AND SURFACE CLEANING APPARATUS
INCLUDING THE SAME**

FIELD

This application relates to the field of cyclonic air treatment members and surface cleaning apparatus including the same.

INTRODUCTION

The following is not an admission that anything discussed below is part of the prior art or part of the common general knowledge of a person skilled in the art.

Various types of surface cleaning apparatus are known, including upright surface cleaning apparatus, canister surface cleaning apparatus, stick surface cleaning apparatus, central vacuum systems, and hand carryable surface cleaning apparatus such as hand vacuums. Further, various designs for cyclonic hand vacuum cleaners, including battery operated cyclonic hand vacuum cleaners, are known in the art.

Surface cleaning apparatus are known which utilize one or more cyclones. A cyclone has a dirt collection region. The dirt collection region may be internal of the cyclone chamber (e.g., the dirt collection region may be a lower end of the cyclone chamber. Alternately, the dirt collection region may be a separate dirt collection chamber that is external to the cyclone chamber and in communication with the cyclone chamber via a dirt outlet. The dirt out may be a slot formed in the sidewall of a cyclone chamber or a gap provided between the end of the cyclone wall and an end of the cyclone chamber.

SUMMARY

In accordance with one aspect of this disclosure, a cyclone chamber is provided with a dirt collection chamber that is in communication with the cyclone chamber by two or more dirt outlet regions. The two dirt outlet regions may be discrete outlets (i.e., each dirt outlet region may be a dirt outlet that is surrounded by, e.g., a portion of the sidewall of the cyclone chamber or a portion of the sidewall of the cyclone chamber and a portion of an end wall of the cyclone chamber) or they may be contiguous (e.g., they may be connected by a gap or slot formed in the cyclone chamber sidewall so as to form a single dirt outlet opening in, e.g., the cyclone chamber sidewall).

An advantage of this design is that dirt which is separated from the air swirling in the cyclone chamber prior to the swirling air reaching an end of the cyclone chamber opposed to the cyclone air inlet end (e.g., after the air has turned, for example, 1 or 2 times in the cyclone chamber) may be removed from the cyclone chamber by a first dirt outlet region and the remainder of the dirt may be separated in a second dirt outlet region that is located closer to or at the end of the cyclone chamber opposed to the cyclone air inlet end.

In accordance with this aspect, there is provided a cyclonic air treatment member comprising:

- (a) a cyclone having a cyclone sidewall, a cyclone first end, an opposed cyclone second end, a cyclone air inlet proximate the cyclone first end, a cyclone air outlet and a cyclone longitudinal axis extending from the cyclone first end to the cyclone second end, wherein a cyclone chamber is located between the cyclone first and second ends and the cyclone chamber has an outer perim-

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eter which comprises the cyclone sidewall, wherein an air flow path extends from the cyclone air inlet to the cyclone air outlet: and,

- (b) a dirt collection chamber external to the cyclone chamber, the dirt collection chamber having first and second dirt outlet regions, each dirt outlet region extending around a portion of the perimeter of the cyclone chamber, wherein the second dirt outlet region is positioned proximate the cyclone second end, and the first dirt outlet region is positioned toward the cyclone first end relative to the second dirt outlet region.

In any embodiment, the first dirt outlet region may be longitudinally spaced apart from and discrete from the second dirt outlet region.

In any embodiment, the second dirt outlet region may be longitudinally spaced apart from and contiguous with the first dirt outlet region.

In any embodiment, the first dirt outlet region may be angularly offset about the outer perimeter of the cyclone chamber as compared to the second dirt outlet region.

In any embodiment, at least one of the first and second dirt outlet regions may comprise a slot extending angularly around a portion of the perimeter of the cyclone chamber.

In any embodiment, at least one of the first and second dirt outlet regions may comprise an array of 4 or more (e.g., 4, 5, 6, 7, 8, 9 or 10) apertures formed in the cyclone sidewall.

In any embodiment, the first dirt outlet region may comprise a slot formed in the cyclone sidewall, and the second dirt outlet region comprises an array of 4 or more (e.g., 4, 5, 6, 7, 8, 9 or 10) apertures formed in the cyclone sidewall and positioned adjacent the first dirt outlet region between the cyclone first end and the first dirt outlet region.

In any embodiment, each of the first and second dirt outlet regions may have a long dimension, and the long dimension of the first dirt outlet region is oriented generally transverse to the long dimension of the second dirt outlet region.

In any embodiment, the air flow path may include a cyclonic path portion that extends cyclonically from the cyclone air inlet toward the cyclone second end, and at least one of the dirt outlet regions may have a long dimension that is aligned with the cyclonic path portion. At least 75% of the first dirt outlet region may extend along a portion of the cyclonic path portion. Alternately, the first dirt outlet region may extend along the cyclonic path from an upstream outlet end of the first dirt outlet region to a downstream outlet end of the first dirt outlet region.

In any embodiment, the downstream outlet end of the first dirt outlet region may be positioned towards the cyclone second end relative to the upstream outlet end of the first dirt outlet region.

In any embodiment, both of the upstream outlet end of the first dirt outlet region and the downstream outlet end of the first dirt outlet region may be located along a portion of the cyclonic path portion.

In any embodiment, the second dirt outlet region may have a long dimension having a radial projection that is aligned perpendicularly to the cyclone axis. Alternately or in addition, the first dirt outlet region may have a long dimension having a radial projection that is aligned parallel to the cyclone axis.

In any embodiment, the second dirt outlet region may be bordered by the cyclone second end.

In any embodiment, the cyclone may further comprise a third dirt outlet region to the dirt collection chamber, the third dirt outlet region is formed in the cyclone sidewall, and is oriented transverse to the first and second dirt outlet regions. The first, second, and third dirt outlet regions may

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be contiguous. Alternately one, two or all three may be discrete or one may be discrete and two may be contiguous.

In any embodiment, the cyclone air outlet may be at the cyclone second end. Alternately, the cyclone air outlet may be at the cyclone first end.

In accordance with this aspect, there is also provided a surface cleaning apparatus comprising the any embodiment of the cyclonic air treatment member disclosed herein.

BRIEF DESCRIPTION OF THE DRAWINGS

For a better understanding of the described embodiments and to show more clearly how they may be carried into effect, reference will now be made, by way of example, to the accompanying drawings in which:

FIG. 1 is a perspective view of a surface cleaning apparatus in accordance with an embodiment;

FIG. 2 is a cross-sectional view taken along line 2-2 in FIG. 1, in accordance with an embodiment;

FIG. 3 is a perspective view of an air treatment member of the apparatus of FIG. 1 with a front wall and air outlet passage omitted, in accordance with an embodiment;

FIG. 4 is a perspective view of the air treatment member of the apparatus of FIG. 1, sectioned along line 2-2 in FIG. 1, and with the front wall and air outlet passage omitted, in accordance with the embodiment of FIG. 3;

FIG. 5 is a perspective view of the air treatment member of the apparatus of FIG. 1, sectioned along line 5-5 in FIG. 1, and with the front wall and air outlet passage omitted, in accordance with the embodiment of FIG. 3;

FIG. 6 is a perspective view of an alternate embodiment of the air treatment member of the apparatus of FIG. 1 with the front wall and air outlet passage omitted, in accordance with another embodiment;

FIG. 7 is a perspective view of the alternate air treatment member of FIG. 6, sectioned along line 2-2 in FIG. 1, and with the front wall and air outlet passage omitted, in accordance with the embodiment of FIG. 6;

FIGS. 8-21 are perspective views of the air treatment member of the apparatus of FIG. 1, sectioned along line 5-5 in FIG. 1, and with the front wall and air outlet passage omitted, in accordance with various embodiments;

FIG. 22 is a cross-sectional view taken along line 2-2 in FIG. 1, in accordance with another embodiment;

FIG. 23 is a cross-sectional view taken along line 2-2 in FIG. 1, in accordance with another embodiment;

FIG. 24 is a perspective view of an upright surface cleaning apparatus in accordance with an embodiment;

FIG. 25 is a cross-sectional view taken along line 25-25 in FIG. 24, in accordance with another embodiment;

FIG. 26 is a perspective view of the surface cleaning apparatus of claim 1 sectioned along line 2-2, in accordance with another embodiment; and,

FIG. 27 is a perspective view of the surface cleaning apparatus of claim 1 sectioned along line 27-27, in accordance with another embodiment.

DESCRIPTION OF EXAMPLE EMBODIMENTS

Numerous embodiments are described in this application, and are presented for illustrative purposes only. The described embodiments are not intended to be limiting in any sense. The invention is widely applicable to numerous embodiments, as is readily apparent from the disclosure herein. Those skilled in the art will recognize that the present invention may be practiced with modification and alteration without departing from the teachings disclosed herein.

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Although particular features of the present invention may be described with reference to one or more particular embodiments or figures, it should be understood that such features are not limited to usage in the one or more particular embodiments or figures with reference to which they are described.

The terms “an embodiment,” “embodiment,” “embodiments,” “the embodiment,” “the embodiments,” “one or more embodiments,” “some embodiments,” and “one embodiment” mean “one or more (but not all) embodiments of the present invention(s),” unless expressly specified otherwise.

The terms “including,” “comprising” and variations thereof mean “including but not limited to,” unless expressly specified otherwise. A listing of items does not imply that any or all of the items are mutually exclusive, unless expressly specified otherwise. The terms “a,” “an” and “the” mean “one or more,” unless expressly specified otherwise.

As used herein and in the claims, two or more parts are said to be “coupled”, “connected”, “attached”, “joined”, “affixed”, or “fastened” where the parts are joined or operate together either directly or indirectly (i.e., through one or more intermediate parts), so long as a link occurs. As used herein and in the claims, two or more parts are said to be “directly coupled”, “directly connected”, “directly attached”, “directly joined”, “directly affixed”, or “directly fastened” where the parts are connected in physical contact with each other. As used herein, two or more parts are said to be “rigidly coupled”, “rigidly connected”, “rigidly attached”, “rigidly joined”, “rigidly affixed”, or “rigidly fastened” where the parts are coupled so as to move as one while maintaining a constant orientation relative to each other. None of the terms “coupled”, “connected”, “attached”, “joined”, “affixed”, and “fastened” distinguish the manner in which two or more parts are joined together.

Further, although method steps may be described (in the disclosure and/or in the claims) in a sequential order, such methods may be configured to work in alternate orders. In other words, any sequence or order of steps that may be described does not necessarily indicate a requirement that the steps be performed in that order. The steps of methods described herein may be performed in any order that is practical. Further, some steps may be performed simultaneously.

As used herein and in the claims, two elements are said to be “parallel” where those elements are parallel and spaced apart, or where those elements are collinear.

Some elements herein may be identified by a part number, which is composed of a base number followed by an alphabetical or subscript-numerical suffix (e.g. 112a, or 112₁). Multiple elements herein may be identified by part numbers that share a base number in common and that differ by their suffixes (e.g. 112₁, 112₂, and 112₃). All elements with a common base number may be referred to collectively or generically using the base number without a suffix (e.g. 112).

General Description of a Hand Vacuum Cleaner

Referring to FIGS. 1-2, an exemplary embodiment of a surface cleaning apparatus is shown generally as 100. The following is a general discussion of apparatus 100 which provides a basis for understanding several of the features which are discussed herein. As discussed subsequently, each of the features may be used individually or in any particular combination or sub-combination in this or in other embodiments disclosed herein.

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Embodiments described herein include an improved cyclonic air treatment member **116**, and a surface cleaning apparatus **100** including the same. Surface cleaning apparatus **100** may be any type of surface cleaning apparatus, including for example a hand vacuum cleaner as shown, a stick vacuum cleaner, an upright vacuum cleaner (**100** in FIG. **24**), a canister vacuum cleaner, an extractor, or a wet/dry type vacuum cleaner.

In FIGS. **1-2**, surface cleaning apparatus **100** is illustrated as a hand vacuum cleaner, which may also be referred to also as a “handvac” or “hand-held vacuum cleaner”. As used herein, a hand vacuum cleaner is a vacuum cleaner that can be operated to clean a surface generally one-handedly. That is, the entire weight of the vacuum may be held by the same one hand used to direct a dirty air inlet of the vacuum cleaner with respect to a surface to be cleaned. For example, handle **104** and dirty air inlet **108** may be rigidly coupled to each other (directly or indirectly), such as being integrally formed or separately molded and then non-removably secured together (e.g. adhesive or welding), so as to move as one while maintaining a constant orientation relative to each other. This is to be contrasted with canister and upright vacuum cleaners, whose weight is typically supported by a surface (e.g. a floor) during use. When a canister vacuum cleaner is operated, or when an upright vacuum cleaner is operated in a ‘lift-away’ configuration, a second hand is typically required to direct the dirty air inlet at the end of a flexible hose.

Still referring to FIGS. **1-2**, surface cleaning apparatus **100** includes a main body or a handvac body **112** having an air treatment member **116** (which may be permanently affixed to the main body or may be removable in part or in whole therefrom for emptying), a dirty air inlet **108**, a clean air outlet **120**, and an air flow path **124** extending between the dirty air inlet **108** and the clean air outlet **120**.

Surface cleaning apparatus **100** has a front end **128**, a rear end **132**, an upper end (also referred to as the top) **136**, and a lower end (also referred to as the bottom) **140**. In the embodiment shown, dirty air inlet **108** is at an upper portion of apparatus front end **128** and clean air outlet **120** is at a rearward portion of apparatus **100** at apparatus rear end **132**. It will be appreciated that dirty air inlet **108** and clean air outlet **120** may be positioned in different locations of apparatus **100**.

A suction motor **144** is provided to generate vacuum suction through air flow path **124**, and is positioned within a motor housing **148**. Suction motor **144** may be a fan-motor assembly including an electric motor and impeller blade(s). In the illustrated embodiment, suction motor **144** is positioned in the air flow path **124** downstream of air treatment member **116**. In this configuration, suction motor **144** may be referred to as a “clean air motor”. Alternatively, suction motor **144** may be positioned upstream of air treatment member **116**, and referred to as a “dirty air motor”.

Air treatment member **116** is configured to remove particles of dirt and other debris from the air flow. In the illustrated example, air treatment member **116** includes a cyclone assembly (also referred to as a “cyclone bin assembly”) having a single cyclonic cleaning stage with a single cyclone **152** and a dirt collection chamber **156** (also referred to as a “dirt collection region”, “dirt collection bin”, “dirt bin”, or “dirt chamber”). Cyclone **152** has a cyclone chamber **154**. Dirt collection chamber **156** may be external to the cyclone chamber **154** (i.e. dirt collection chamber **156** may have a discrete volume from that of cyclone chamber **154**). Cyclone **152** and dirt collection chamber **156** may be of any configuration suitable for separating dirt from an air stream

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and collecting the separated dirt respectively, and may be in communication dirt outlet(s) of the cyclone chamber.

In alternate embodiments, air treatment member **116** may include a cyclone assembly having two or more cyclonic cleaning stages arranged in series with each other. Each cyclonic cleaning stage may include one or more cyclones arranged in parallel with each other and one or more dirt collection chambers, of any suitable configuration. The dirt collection chamber(s) may be external to the cyclone chambers of the cyclones. Each cyclone may have its own dirt collection chamber or two or more cyclones fluidically connected in parallel may have a single common dirt collection chamber.

Referring to FIG. **2**, hand vacuum cleaner **100** may include a pre-motor filter **160** provided in the air flow path **124** downstream of air treatment member **116** and upstream of suction motor **144**. Pre-motor filter **160** may be formed from any suitable physical, porous filter media. For example, pre-motor filter **160** may be one or more of a foam filter, felt filter, HEPA filter, or other physical filter media. In some embodiments, pre-motor filter **160** may include an electrostatic filter, or the like. As shown, pre-motor filter **160** may be located in a pre-motor filter housing **164** that is external to the air treatment member **116**.

In the illustrated embodiment, dirty air inlet **108** is the inlet end **168** of an air inlet conduit **172**. Optionally, inlet end **168** of air inlet conduit **172** can be used as a nozzle to directly clean a surface. Alternatively, or in addition to functioning as a nozzle, air inlet conduit **172** may be connected (e.g. directly connected) to the downstream end of any suitable accessory tool such as a rigid air flow conduit (e.g., an above floor cleaning wand), a crevice tool, a mini brush, and the like. As shown, dirty air inlet **108** may be positioned forward of air treatment member **116**, although this need not be the case.

In the embodiment of FIG. **2**, the air treatment member **116** comprises a cyclone **152**, the air treatment air inlet is a cyclone air inlet **184**, and the air treatment member air outlet is a cyclone air outlet **188**. Accordingly, in operation, after activating suction motor **144**, dirty air enters apparatus **100** through dirty air inlet **108** and is directed along air inlet conduit **172** to the cyclone air inlet **184**. As shown, cyclone air inlet **184** may direct the dirty air flow to enter cyclone chamber **154** in a tangential direction so as to promote cyclonic action. Dirt particles and other debris may be disentrained (i.e. separated) from the dirty air flow as the dirty air flow travels from cyclone air inlet **184** to cyclone air outlet **188**. The disentrained dirt particles and debris may discharge from cyclone chamber **154** through a dirt outlet **190** into dirt collection chamber **156** external to the cyclone chamber **154**, where the dirt particles and debris may be collected and stored until dirt collection chamber **156** is emptied.

Air exiting cyclone chamber **154** may pass through an outlet passage **192** located upstream of cyclone air outlet **188**. Cyclone chamber outlet passage **192** may also act as a vortex finder to promote cyclonic flow within cyclone chamber **154**. In some embodiments, cyclone outlet passage **192** may include a screen or shroud **196** (e.g. a fine mesh screen) in the air flow path **124** to remove large dirt particles and debris, such as hair, remaining in the exiting air flow.

From cyclone air outlet **188**, the air flow may be directed into pre-motor filter housing **164** at an upstream side **196** of pre-motor filter **160**. The air flow may pass through pre-motor filter **160**, and then exit through pre-motor filter chamber air outlet **198** into motor housing **148**. At motor housing **148**, the clean air flow may be drawn into suction

motor **144** and then discharged from apparatus **100** through clean air outlet **120**. Prior to exiting the clean air outlet **120**, the treated air may pass through a post-motor filter **176**, which may be one or more layers of filter media.

Power may be supplied to suction motor **144** and other electrical components of apparatus **100** from an onboard energy storage member which may include, for example, one or more batteries or other energy storage device. In the illustrated embodiment, apparatus **100** includes a battery pack **180**. Battery pack **180** may be permanently connected to apparatus **100** and rechargeable in-situ, or removable from apparatus **100**. In the example shown, battery pack **180** is located between handle **104** and air treatment member **116**. Alternatively or in addition to battery pack **180**, power may be supplied to apparatus **100** by an electrical cord (not shown) connected to apparatus **100** that can be electrically connected to mains power by at a standard wall electrical outlet.

Cyclonic Air Treatment Member

Embodiments herein relate to an improved cyclonic air treatment member. The features in this section may be used by themselves in any surface cleaning apparatus or in any combination or sub-combination with any other feature or features described herein.

Within a cyclone, dirt is disentrained from a dirt laden air flow by directing the air flow along a cyclonic path. The cyclonic flow direction imparts radially outward forces upon dirt particles in the air flow, whereby the dirt particles are separated from the air flow and ultimately, e.g. ride against the cyclone sidewall. Dirt moved against the cyclone sidewall may exit from the cyclone chamber to a dirt collection chamber through a dirt outlet.

The ability of a cyclonic flow to separate dirt particles depends in part on the radial acceleration experienced by the dirt particles as a result of their cyclonic velocity through the cyclone. However, the cyclonic particle velocity may slow between the cyclone air inlet and air outlet. Below a threshold cyclonic particle velocity, the separation efficiency (i.e. the percentage of dirt particles separated from the dirty air flow by the cyclone) may be substantially reduced. When a vacuum cleaner operates at a high air flow rate (e.g. a 'high power mode' in a handvac), the cyclonic particle velocity between the cyclone air inlet and air outlet may remain well above such threshold velocity. However, when a vacuum cleaner operates at a low air flow rate (e.g. a 'low power mode' in a handvac), the cyclonic particle velocity may fall below the threshold velocity at some point between the cyclone air inlet and air outlet. In such a case, some of the dirt particles that have already been disentrained may be reentrained.

Embodiments herein relate to an improved cyclone having at least one additional dirt outlet region that may be positioned closer, along the cyclonic air flow path, to the cyclone air inlet. The additional dirt outlet region may be positioned at a location at which the cyclonic particle velocity may still be high enough (e.g. above the threshold velocity) to provide a targeted separation efficiency, even when operating at a lower air flow rate. Thus, the additional dirt outlet may permit the apparatus to optionally operate at a lower air flow rate with less loss of separation efficiency, all else being equal. For a handvac, this may mitigate the loss of separation efficiency when operating in a 'low power mode', which otherwise has an advantage of consuming less power thereby providing a longer run-time on a single charge.

Referring to FIGS. 2-4, cyclone **152** includes a cyclone sidewall **202** that, as exemplified, extends along a cyclone longitudinal axis **204** between a cyclone first end **206** and a cyclone second end **208**. Accordingly, cyclone chamber **154** is bounded by cyclone sidewall **202** and cyclone first and second ends **206**, **208**. Cyclone **152** includes a tangential air inlet **184**, although any air inlet may be used. As shown, air inlet **184** may be located proximate cyclone first end **206**, although the cyclone air inlet may be provided at other locations. Cyclone also includes an air outlet **188**. Cyclone air outlet **188** may be located proximate cyclone second end **208**, such as in the illustrated uniflow cyclone configuration, or it may be located at cyclone first end **206** (see, for example FIGS. 24-25). Apparatus air flow path **124** includes a cyclone air flow path **212**, which extends from cyclone air inlet **184** to cyclone air outlet **188**.

Referring to FIGS. 3-4, cyclone **152** may include first and second dirt outlet regions **190₁** and **190₂**. Second dirt outlet region **190₂** may be located proximate (e.g. at or closer to) cyclone second end **208**. For example, second dirt outlet region **190₂** may be located at the cyclone second end **208** as exemplified in FIGS. 2 and 3. Second dirt outlet region **190₂** may be of any design known in the vacuum cleaner arts. For example, it may be a slot formed in the cyclone sidewall at the cyclone second end **208** as exemplified or it may be defined by a gap between the cyclone chamber sidewall and the second end wall **208** (e.g., it may be an annular opening at the end of the cyclone sidewall that faces the cyclone second end **208**. First dirt outlet region **190₁** may be located axially or longitudinally towards cyclone first end **206** relative to second dirt outlet region **190₂**.

Referring to FIGS. 4-5, first dirt outlet region **190₁** may be provided anywhere in cyclone sidewall **202** having a longitudinal position between cyclone first end **206** and second dirt outlet **190₂**. For example, first dirt outlet region **190₁** may be longitudinally positioned between cyclone air inlet **184** and second dirt outlet **190₂**. This may allow dirt that enters cyclone **152** to exit through cyclone dirt outlet region **190₁** while that dirt has sufficient cyclonic velocity and before that dirt would have reached second dirt outlet region **190₂**.

In some embodiments, first dirt outlet region **190₁** may be aligned with a cyclonic portion of cyclone air flow path **212** (see for example FIG. 15). This allows separated dirt that is sliding on cyclone sidewall **202** as it is carried along a cyclonic portion of air flow path **212** to flow into first dirt outlet region **190₁**, through which the dirt can exit into dirt collection chamber **156**. Accordingly, the alignment of first dirt outlet region **190₁** may permit the dirt outlet region **190₁** to better interact with dirt separated during an upstream portion of the cyclone air flow path **212**. Even when operating at a low air flow rate, the upstream portion of flow path **212** may yet have sufficient dirt particle velocity to provide a high separation efficiency.

Still referring to FIGS. 4-5, cyclone air flow path **212** may have an axial flow width **216** (i.e. measured parallel to longitudinal axis **204**) approximately equal to an axial width **220** (i.e. measured parallel to longitudinal axis **204**) of cyclone air inlet **184**. Axial flow width **216** may remain generally constant between cyclone air inlet **184** and cyclone second end **208**. Cyclone dirt outlet regions **190** may have any axial width **224** suitable for allowing dirt separated from the air flow to exit cyclone chamber **154** towards dirt collection chamber **156**. Preferably, axial dirt outlet width **224₁** (or axial width **224** of each dirt outlet region **190**) is between 35% and 90% of axial air inlet width **220** (i.e. about 35% to 90% of axial air flow path width **216**). A width **224**

within this range may be large enough to permit common dirt particle sizes to exit freely through the cyclone dirt outlet region **190**, and yet may not be so large that a detrimental amount of the air flow is diverted from cyclone chamber **154** through cyclone dirt outlet region **190**.

In other embodiments, axial dirt outlet width **224₁** may be between 15% and 150% of axial air inlet width **220** (i.e. about 15% to 150% of axial air flow path width **216**), between 25% and 125%, between 40% and 75% or between 50% and 60%. The lower portion of this range (e.g., 10% to 50% or 15% to 35% of axial air inlet width **220**) may minimize the amount of the air flow that diverts through cyclone dirt outlet **190** while still permitting at least small dirt particles to exit. The upper portion of this range (e.g., 75% to 150%, 90% to 150% or 100% to 125% of axial air inlet width **220**) may allow very large dirt particles to exit, although a somewhat greater amount of air flow may divert through cyclone dirt outlet region **190**.

It will be appreciated that first and second dirt outlet regions **190₁** and **190₂** may have the same size (e.g. width, length, and/or area) or may be differently sized.

Alternatively or in addition, the alignment of first dirt outlet region **190₁** with a cyclonic portion of cyclone air flow path **212** may be such that at least 50%, 60%, 70%, 80%, 90% or more of the area of first dirt outlet region **190₁** is coincident with (e.g., extends continuously along) the cyclone air flow path **212**. This may expose separated dirt particles to first dirt outlet region **190₁** for an extended continuous distance along cyclone air flow path **212**, whereby the dirt particles may be more likely to exit through first dirt outlet **190₁**, all else being equal.

The alignment of first dirt outlet region **190₁** with the cyclone air flow path **212** may be such that both an upstream end **228** of dirt outlet region **190₁** and a downstream end **232** of dirt outlet region **190₁** are each located along a portion of the cyclone air flow path **212**. For example, dirt outlet region **190₁** may extend contiguously along a part of the cyclone air flow path **212** from dirt outlet upstream end **228** to dirt outlet downstream end **232**.

Referring to FIG. 4, first dirt outlet region **190₁** may have any axial position (i.e. with respect to cyclone longitudinal axis **204**) between cyclone first end **206** and second dirt outlet **190₂**. In some embodiments, first dirt outlet region **190₁** is axially offset from cyclone air inlet **184** by a distance **236** sufficient to permit at least some dirt particles within the air flow to separate (i.e. move outwardly to the cyclone sidewall **202**) as a result of the cyclonic character of air flow path **212**. For example, first dirt outlet region **190₁** may be located at least one turn (i.e., a 360 degree segment) of cyclone air flow path **212** from cyclone air inlet **184**. In the illustrated example, first dirt outlet region **190₁** is located just under 1.5 turns of cyclone air flow path **212** from cyclone air inlet **184**. Characterized another way, axial distance **236** from cyclone air inlet **184** to dirt outlet upstream end **228**, measured center-to-center may be at least equal to cyclone air inlet width **220** (i.e. at least about cyclone air flow width **216**). More generally, cyclone air inlet **184** may be spaced (center-to-center) from cyclone first end **206** by an axial distance **240** at least equal to cyclone air inlet width **220**.

Cyclone dirt outlet region **190₁** may have any angular (i.e. circumferential) position on cyclone sidewall **202**. In some embodiments, cyclone dirt outlet region **190₁** is angularly located at a bottom end **244** of cyclone sidewall **202** as shown. This allows gravity to assist with moving separated dirt particles through cyclone dirt outlet **190₁**. In other embodiments, cyclone dirt outlet region **190₁** may be angu-

larly offset from sidewall bottom end **244**. Although such positions may not benefit from gravity assistance for discharging separated dirt particles, they may advantageously provide greater flexibility to position cyclone dirt outlet region **190₁** at a distance **252** along cyclone air flow path **212**, at which cyclonic particle velocities and residency time are optimized for separation efficiency (e.g. at the power mode(s) provided by apparatus **100**). As an example, FIGS. 6-7 show cyclone dirt outlet region **190₁** angularly located between sidewall top and bottom ends **248**, **244**. In the example shown, cyclone dirt outlet region **190₁** has a path distance **252** of about one turn (e.g. 360 degrees) from cyclone air inlet **184**.

Referring to FIG. 5, cyclone dirt outlets **190** may have any orientation that is suitable for allowing dirt particles to exit cyclone chamber **154**. For example, one of cyclone dirt outlets region **190** (or both as shown) may be oriented such that they have a radial projection **256** (i.e. onto a plane **260** that includes cyclone longitudinal axis **204**) wherein the long direction is oriented transverse (e.g. perpendicular) to cyclone longitudinal axis **204**. For example, a cyclone dirt outlet region **190** may have a projected axis **264** that is transverse (e.g. perpendicular) to longitudinal axis **204**. As shown in FIG. 4, this may permit cyclone dirt outlet(s) region **190** to be oriented in alignment with cyclone air flow path **212**.

FIG. 5 shows an example in which projections **256** (and projected axes **264**) are substantially perpendicular to cyclone longitudinal axis **204**. FIGS. 8-9 show an example in which projections **256** (and projected axes **264**) are transverse to cyclone longitudinal axis **204** but not perpendicular. For example, projected axes **264** may be up to 30 degrees from perpendicular with longitudinal axis **204**.

FIG. 8 shows dirt outlet regions **190** having a helical orientation, which may be aligned with the cyclonic air flow path through cyclone chamber **154**. As shown, each dirt outlet region **190** has an upstream end **228** located towards cyclone first end **206** relative to its downstream end **232**. An advantage of this design is that it can allow a greater portion of the area of dirt outlet region regions **190** to extend continuously along a portion of the cyclonic air flow path in cyclone chamber **154**.

FIG. 9 shows dirt outlet regions **190** having a helical orientation, which may be transverse (e.g. opposed to, misaligned, or counter-aligned) with the cyclonic air flow path through cyclone chamber **154**. For example, if the cyclonic air flow path **212** from cyclone air inlet **184** is counterclockwise when viewed from cyclone first end **206** looking towards cyclone second end **208** as illustrated in FIG. 4, then one or both of dirt outlet regions **190** may extend clockwise from their outlet upstream end **228** to their outlet downstream end **232** as seen in FIG. 9 (or vice versa). An advantage of a transversely oriented dirt outlet **190** is that it may intersect several turns of the cyclone air flow path which may expose the dirt outlet **190** to dirt particles having a wider range of residency time and particle velocities in the cyclonic flow. This may allow particles of different sizes sufficient time to separate from the air flow and make contact with cyclone sidewall **202**. This design may also permit the dirt outlet region **190** to provide an effective exit for a wider range of air flow rates. Further, where the air flow path within cyclone **152** reverses direction at cyclone second end **208** to travel towards cyclone air outlet **188** (e.g. through cyclone chamber outlet passage **192**, see FIG. 2) this design may align the dirt outlet region **190** with the reversed portion of the air flow path (i.e. the 'counter-flow' portion of the air flow path).

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FIGS. 10 and 11 illustrate examples in which dirt outlet region 190₁ is oriented differently from dirt outlet region 190₂. As shown, one of dirt outlet regions 190 may have a radial projection 256 (and projected axis 264) that is substantially perpendicular to cyclone longitudinal axis 204, and one of dirt outlet regions 190 may have a radial projection 256 (and projected axis 264) that is transverse but not perpendicular to longitudinal axis 204. The illustrated examples show second dirt outlet region 190₂ having a radial projection 256₂ (and projected axis 264₂) that is substantially perpendicular to cyclone longitudinal axis 204, and first dirt outlet region 190₁ having a helical orientation. An advantage of this design is that it allows first dirt outlet region 190₁ to be positioned and oriented to provide an effective dirt outlet for lower air flow rates, while second dirt outlet region 190₂ is bordered by cyclone second end 208 for discharging dirt that passes first dirt outlet region 190₁ and piles against cyclone second end 208. In FIG. 10, first dirt outlet region 190₁ is illustrated with a helical orientation aligned with the cyclonic air flow path through cyclone chamber 154. In FIG. 11, first dirt outlet region 190₂ is illustrated with a helical orientation that is transverse (e.g. opposed, misaligned, or counter-aligned) to the cyclonic air flow path through cyclone chamber 154.

Reference is now made to FIG. 12. In some embodiments, first dirt outlet region 190₁ may have a long direction that may be oriented substantially parallel (e.g. within 15 degrees of parallel) with cyclone longitudinal axis 204. An advantage of this design it that is can allow first dirt outlet region 190₁ to intersect several turns of the cyclone air flow path. This allows dirt outlet region 190₁ to provide an exit for dirt particles that have experienced a wider range of residency time and particle velocities in the cyclonic flow. In turn, this may allow particles of different sizes sufficient time to separate from the air flow and make contact with cyclone sidewall 202. This design may also permit the dirt outlet region 190 to provide an effective dirt outlet for a wider range of air flow rates. As shown, first dirt outlet region 190₁ may have a radial projection 256₁ (and projected axis 264₁) that is parallel to cyclone longitudinal axis 204.

FIG. 13 shows an embodiment in which the long direction of first dirt outlet region 190₁ has an orientation that is between a transverse and a parallel orientation relative to cyclone longitudinal axis 204. Such an orientation may provide a balance between (i) providing some degree of alignment with the cyclonic air flow path through cyclone chamber 154 in one of the forward direction (i.e. from cyclone first end 206 towards cyclone second end 208) or the reverse direction (i.e. from cyclone second end 208 towards cyclone first end 206), and (ii) exposing the dirt outlet 190₂ to several turns of the cyclonic air flow path.

Reference is now made to FIGS. 14-16. As shown, some embodiments of cyclone 152 may have first dirt outlet region 190₁ contiguous with second dirt outlet 190₂. Accordingly, as opposed to, e.g., FIG. 13 wherein two discrete outlet slots are provided, a single outlet slot or opening or gap in the sidewall may be provided which comprises two or more dirt outlet regions. An advantage of this design is that it may provide, where the first and second dirt outlet regions 190₁ and 190₂ meet, an outlet region having a large outlet width and length, which can accommodate especially large dirt particles. In the illustrated example, the first and second dirt outlet regions 190₁ and 190₂ have different orientations relative to cyclone longitudinal axis 204. As shown, first dirt outlet region 190₁ may have a downstream end 232 that is connected to second dirt outlet region 190₂. Downstream end 232 may be positioned towards cyclone second end 208

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relative to cyclone first end 206. This may provide the combination of dirt outlet regions 190₁ and 190₂ with a "T-shape" configuration. As shown in FIG. 14, first dirt outlet region 190₁ may be oriented substantially parallel to cyclone longitudinal axis 204. As shown in FIGS. 15-16, first dirt outlet region 190₁ may have a curved shape that is oriented neither parallel nor perpendicular to cyclone longitudinal axis 204.

Referring to FIGS. 17-19, cyclone 152 may have three dirt outlet regions 190 in some embodiments. As shown, third dirt outlet region 190₃ may be oriented transverse to first and second dirt outlet regions 190₁ and 190₂. First and second dirt outlet regions 190₁ and 190₂ may be oriented the same (as shown), or differently from each other. An advantage of this design is that it may permit (i) first dirt outlet region 190₁ to be oriented best to provide an exit for dirt particles when operating at low air flow rates, (ii) second dirt outlet region 190₂ to provide an exit for particles that reach cyclone second end 208, and (iii) third dirt outlet region 190₃ to interact with several turns of the cyclonic air flow path, which as discussed above may provide an exit for dirt particles that have experienced a wider range of residency time and particle velocities in the cyclonic flow, allow particles of different sizes sufficient time to separate from the air flow and make contact with cyclone sidewall, and/or provide an effective dirt outlet for a wider range of air flow rates.

As shown, the combination of dirt outlet regions 190₁, 190₂, 190₃ may have an "H-shape" or "N-shape" configuration. In the illustrated embodiment, third dirt outlet region 190₃ is contiguous with first and second dirt outlets 190₁ and 190₂. As exemplified, third dirt outlet 190₃ has an upstream end 228₃ connected to first dirt outlet region 190₁, and a downstream end 232₃ connected to second dirt outlet region 190₂. In alternative embodiments, third dirt outlet region 190₃ may be spaced apart from (e.g. discontinuous with) one or both of first and second dirt outlet regions 190₁, 190₂ such that two or 3 discrete outlets are provided. FIG. 17 shows an example in which third dirt outlet region 190₃ is oriented parallel to cyclone longitudinal axis 204. FIGS. 18-19 show examples in which third dirt outlet region 190₃ is oriented non-parallel to cyclone longitudinal axis 204 (e.g. neither perpendicular nor parallel to cyclone longitudinal axis 204, as shown).

In other embodiments, first dirt outlet region 190₁ may be spaced apart from (e.g. discontinuous with) second dirt outlet 190₂, as illustrated in the examples of FIGS. 3-13.

Referring to FIG. 4, any or all of dirt outlet regions 190 may be formed in cyclone sidewall 202. For example, a dirt outlet 190 may include an aperture (e.g. hole or slot) in cyclone sidewall 202 that allows separated dirt particles to exit cyclone chamber 154 towards dirt collection chamber 154. In the illustrated example, dirt outlet regions 190 are formed in a portion of cyclone sidewall 202 that is common to dirt collection chamber 156. An advantage of this design is that it provides the shortest travel distance from dirt outlet 190 to dirt collection chamber 156, which may mitigate dirt particles collecting in an intervening passage. However, in alternative embodiments dirt outlet region 190 may provide an entrance to a passage leading to dirt collection chamber 156. This may provide greater flexibility in the location of dirt collection chamber 156 relative cyclone chamber 154, such as to optimize apparatus 100 for compactness. Embodiments having a dirt outlet passage are discussed below.

FIG. 4 shows an example in which dirt outlet regions 190 are formed as slots in cyclone sidewall 202 (e.g., an open having a long dimension that extends circumferentially

around a portion of the sidewall). As shown in FIG. 20, a dirt outlet region 190 may be formed as an array of 4 or more closely arranged discrete apertures 268 that collectively define the dirt outlet region 190. As compared to a slot, an array of apertures 268 may provide many smaller apertures that are discontinuous with each other. This may help to reduce the amount of the air flow which diverts into dirt collection chamber 156, which in turn may reduce the backpressure and re-entrainment of collected dirt that can result from such divergence. A dirt outlet region 190 may be composed of an array of 4 or more (e.g., 5, 6, 7, 8, 9 or 10) closely arranged apertures 268 organized in any pattern. In the illustrated embodiment, each dirt outlet region 190 is formed as 4 equally sized apertures 268 arranged linearly in a single row. In other embodiment, each dirt outlet region 190 may be formed from more than 4 apertures, which may be the same or differently sized, and which may be arranged in one or many rows (or in a different non-linear pattern). It is expressly contemplated that any embodiment described or shown herein as a slot may also be formed in another embodiment as an array of apertures.

Referring to FIGS. 21-22, in some embodiments cyclone 152 includes one or more groups 272 of small apertures 274 (e.g. 10 or more apertures 274) adjacent one or more (or all) of dirt outlet regions 190. For example, a group 272 may be located towards cyclone first end 206 relative to the adjacent dirt outlet region 190 (e.g. upstream of the adjacent dirt outlet region 190). Aperture group 272 may provide an exit for small dirt particles which remain open in the event that the adjacent dirt outlet region 190 becomes clogged. As shown, each group 272 may be angularly aligned (e.g. circumferentially aligned) with its respective adjacent dirt outlet region 190. The illustrated embodiment shows a first group 272₁ of apertures adjacent dirt outlet region 190₁ and located between first dirt outlet region 190₁ and cyclone first end 206, and a second group 272₂ of apertures adjacent dirt outlet region 190₂ and located between second dirt outlet region 190₂ and first dirt outlet 190₁. As shown, first group 272₁ may be axially spaced from first end 206 and second group 272₂ may be axially spaced from first dirt outlet 190₁. FIG. 23 shows an alternative embodiment in which second group extends from proximate second dirt outlet region 190₂ to proximate first dirt outlet 190₁.

Returning to FIG. 21, each aperture 274 may have a size (e.g. width, length, and/or area) that is substantially smaller than the associated adjacent dirt outlet region 190. In some embodiments, aperture 274 may have a width 288 of between 0.10 inches to 0.20 inches. This may provide a size that accommodates most small dirt particles collected in domestic (e.g. residential and commercial) environments. More generally, apertures 274 may each have a width 288 of between 0.010 inches and 0.500 inches. Apertures 274 having a width 288 of between 0.010 inches and 0.10 inches may provide exits suitable for very fine particles, and may minimize the amount of the air flow that diverts from the cyclone chamber 154 through apertures 274. Apertures 274 having a width 288 of between 0.20 inches and 0.50 inches may provide exits suitable for relatively larger particles, although somewhat more of the air flow may divert from cyclone chamber 154 through apertures 274. This may provide an acceptable trade-off where the dirt particles targeted for collection by apparatus 100 tend to be larger.

Turning to FIG. 2, cyclone chamber outlet passage 192 may have any shape that can provide an outlet passage for air exiting cyclone chamber 154. Cyclone chamber outlet passage 192 may extend longitudinally from a passage second end 276 at cyclone second end 208 towards cyclone

first end 206 (e.g. in parallel with cyclone longitudinal axis 204) to a passage first end 280. As shown, cyclone chamber outlet passage 192 may be spaced apart from cyclone sidewall 202 to define a surrounding annular region between cyclone chamber outlet passage 192 and cyclone sidewall 202 that promotes cyclonic air flow through cyclone chamber 154.

In the illustrated embodiment, cyclone chamber outlet passage 192 has a transverse width 288 (e.g. diameter) that is substantially constant (e.g. varies by less than 10%) between passage first end 280 and passage second end 276. Depending on the size and shape of cyclone sidewall 202, this may provide the air flow path through cyclone chamber 154 with a relatively constant cross-sectional area.

Turning to FIG. 22, in any embodiment disclosed herein, cyclone chamber outlet passage 192 may have a transverse width 288 that increases between passage first end 280 and passage second end 276 towards passage second end 276. In other words, cyclone chamber outlet passage 192 may taper in transverse width 288 towards passage first end 280. Depending on the size and shape of cyclone sidewall 202, this may provide the air flow path through cyclone chamber 154 with a shrinking cross-sectional area as the air flow travels from cyclone air inlet 184 towards cyclone second end 208. As a result of the inverse relationship between cross-sectional area and velocity, the progressive reduction in cross-sectional flow area may increase the flow velocity towards cyclone second end 208. This may mitigate a loss of velocity and cyclonic degradation that may develop towards cyclone second end 208 particularly when operating at low flow rates (e.g. in a lower power mode). Consequently, the tapered cyclone chamber outlet passage 192 may promote greater overall separation efficiency for cyclone 152.

As shown, transverse width 288 may increase continuously between passage first end 280 and passage second end 276. In some embodiments, transverse width 288 may increase by at least 10% (e.g. by 10% to 200%, 25% to 175%, 40% to 125% or 60% to 90%) between passage first end 280 and passage second end 276. In the illustrated embodiment, transverse width 288 increases by about 125% between passage first end 280 and passage second end 276.

Although many of the figures illustrate concepts and embodiments applied to an exemplary handvac, all of the embodiments described herein apply equally to other surface cleaning apparatus (e.g. upright vacuums, canister vacuums, etc.). Further, although many of the figures illustrate a uniflow cyclone that is horizontally oriented, all embodiments disclosed here are also applicable to other cyclone configurations and orientations. As an example, FIGS. 24-25 show an upright vacuum 100 having a cyclonic air treatment member 116 with an inverted cyclone 152. As shown, cyclone 152 has a central longitudinal axis 204 that is vertically oriented, a plurality of dirt outlet regions 190 (which may have any configuration disclosed in any embodiment herein), a cyclone chamber air outlet passage 192 (which may have any configuration disclosed in any embodiment here), and both the cyclone air inlet 184 and outlet 188 are located at cyclone first end 206.

Reference is now made to FIGS. 26-27. In some embodiments, a dirt outlet region 190 may provide an entryway to a dirt outlet passage 292 leading to dirt collection chamber 156. This may be the case for the only dirt outlet region 190 of a cyclone 152 as shown, or for one or more (or all) dirt outlet regions 190 of a cyclone 152 having many dirt outlet regions 190 (e.g. as in any embodiment disclosed herein having two or more dirt outlets 190). An advantage of providing a dirt outlet passage 292 between a dirt outlet

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region 190 and the dirt collection chamber 156 is that it may reduce the amount of air flow that diverts from the cyclone chamber 154 into the dirt collection chamber 156. Diverted air flow can produce a pressure drop in the air flow through cyclone 152, which may result in less suction and possibly lower dirt separation efficiency all else being equal. By mitigating pressure drops, a smaller, lighter, less expensive suction motor may be used to achieve the same suction, or greater suction may be achieved with the same suction motor. Further, diverted air flow may disturb dirt that has collected in dirt collection chamber 156, which may lead to that dirt re-emerging into the cyclone chamber 154 through the dirt outlet region 190. A dirt outlet passage 292 may help to mitigate dirt collected in dirt collection chamber 156 from returning to cyclone chamber 154.

Dirt outlet passage 292 has a length 296 extending from dirt outlet region 190 to passage outlet 304. Passage outlet 304 may be located inside dirt collection chamber 156 as shown, or may be formed in a sidewall of dirt collection chamber 156 (e.g., the outlet end may be a port provided in a sidewall of the dirt collection chamber 156). Passage outlet 304 may have any passage length 296 suitable for directing dirt exiting from cyclone chamber 154 at a dirt outlet region 190 to dirt collection chamber 156. Preferably, passage length 296 is greater than a thickness of cyclone chamber sidewall 202. For example passage length 296 may be greater than 5 mm (e.g. between 5 mm and 300 mm, 25-250 mm, 50-200 mm or 75-150 mm). A passage length 296 closer to 5 mm may be appropriate where, for example cyclone chamber 154 and dirt collection chamber 156 share a common dividing wall 202. A passage length much greater than 5 mm (e.g. 50 mm or more) may be appropriate where, for example cyclone chamber 154 and dirt collection chamber 156 are spaced apart.

Dirt outlet passage 292 may extend in any direction from dirt outlet region 190 towards dirt collection chamber 156. In some embodiments, dirt outlet passage 292 is oriented tangential to cyclone chamber 154. FIG. 26 shows an example in which dirt outlet passage 292 is oriented tangential cyclone chamber 154 in alignment with the direction of cyclone air flow path 212 where cyclone air flow path 212 crosses dirt outlet region 190. An advantage of this design is that dirt outlet passage 292 may be oriented in the same direction as the direction of dirt particles at dirt outlet 190. This may increase particle separation efficiency by reducing the number of dirt particles which cross over dirt outlet region 190 without exiting cyclone chamber 154. However, such tangential alignment may also lead to a somewhat greater amount of the air flow diverting from cyclone chamber 154 into dirt collection chamber 156. FIG. 27 shows an example in which dirt outlet passage 292 is oriented tangential to cyclone chamber 154 but extending in a direction opposed to the direction of cyclone air flow path 212 where cyclone air flow path 212 crosses dirt outlet 190. An advantage of this design is that it may reduce the amount of air that diverts from cyclone chamber 154 to dirt collection chamber 156, although a somewhat greater number of dirt particles may pass over dirt outlet 190 without exiting.

While the above description provides examples of the embodiments, it will be appreciated that some features and/or functions of the described embodiments are susceptible to modification without departing from the spirit and principles of operation of the described embodiments. Accordingly, what has been described above has been intended to be illustrative of the invention and non-limiting and it will be understood by persons skilled in the art that other variants and modifications may be made without

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departing from the scope of the invention as defined in the claims appended hereto. The scope of the claims should not be limited by the preferred embodiments and examples, but should be given the broadest interpretation consistent with the description as a whole.

The invention claimed is:

1. A cyclonic air treatment member comprising:

(a) a cyclone having a cyclone sidewall, a cyclone first end, an opposed cyclone second end, a cyclone air inlet proximate the cyclone first end, a cyclone air outlet proximate the cyclone second end and a cyclone longitudinal axis extending from the cyclone first end to the cyclone second end, wherein a cyclone chamber is located between the cyclone first and second ends and the cyclone chamber has an outer perimeter which comprises the cyclone sidewall, the cyclone sidewall having a plurality of dirt outlets comprising first and second dirt outlet regions wherein an air flow path extends from the cyclone air inlet to the cyclone air outlet, wherein an inner surface of the cyclone sidewall is a contiguous uninterrupted surface other than the dirt outlet regions;

(b) a dirt collection chamber external to the cyclone chamber; and

(c) a suction motor downstream from the cyclone air outlet whereby air which has exited that cyclone air outlet continues downstream to the suction motor in the absence of reentering the cyclone chamber,

wherein each dirt outlet region extends around a portion of the outer perimeter of the cyclone chamber, wherein the second dirt outlet region is positioned proximate the cyclone second end, and the first dirt outlet region is positioned toward the cyclone first end relative to the second dirt outlet region, and

wherein the first dirt outlet region is discrete from the second dirt outlet region.

2. The cyclonic air treatment member of claim 1, wherein the first dirt outlet region is longitudinally spaced apart from the second dirt outlet region.

3. The cyclonic air treatment member of claim 1, wherein the second dirt outlet region is longitudinally spaced apart from and contiguous with the first dirt outlet region.

4. The cyclonic air treatment member of claim 1, wherein the first dirt outlet region is angularly offset about the outer perimeter of the cyclone chamber as compared to the second dirt outlet region.

5. The cyclonic air treatment member of claim 1, wherein at least one of the first and second dirt outlet regions comprises a slot extending angularly around a portion of the outer perimeter of the cyclone chamber.

6. The cyclonic air treatment member of claim 1, wherein at least one of the first and second dirt outlet regions comprises an array of 4 or more apertures formed in the cyclone sidewall.

7. The cyclonic air treatment member of claim 1, wherein the second dirt outlet region comprises a slot formed in the cyclone sidewall, and the first dirt outlet region comprises an array of 4 or more apertures formed in the cyclone sidewall and positioned adjacent the second dirt outlet region between the cyclone first end and the second dirt outlet region.

8. The cyclonic air treatment member of claim 1, wherein each of the first and second dirt outlet regions has a long dimension, and the long dimension of the first dirt outlet region is oriented generally transverse to the long dimension of the second dirt outlet region.

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9. The cyclonic air treatment member of claim 1, wherein the air flow path includes a cyclonic path portion that extends cyclonically from the cyclone air inlet toward the cyclone second end, and at least one of the dirt outlet regions has a long dimension that is aligned with the cyclonic path portion.

10. The cyclonic air treatment member of claim 9, wherein at least 75% of the first dirt outlet extends along a portion of the cyclonic path portion.

11. The cyclonic air treatment member of claim 9, wherein the first dirt outlet region extends along the cyclonic path from an upstream outlet end of the first dirt outlet region to a downstream outlet end of the first dirt outlet region.

12. The cyclonic air treatment member of claim 11, wherein the downstream outlet end of the first dirt outlet region is positioned towards the cyclone second end relative to the upstream outlet end of the first dirt outlet region.

13. The cyclonic air treatment member of claim 11, wherein both of the upstream outlet end of the first dirt outlet region and the downstream outlet end of the first dirt outlet region are located along a portion of the cyclonic path portion.

14. The cyclonic air treatment member of claim 1, wherein the second dirt outlet region has a long dimension having a radial projection that is aligned perpendicularly to the cyclone longitudinal axis.

15. The cyclonic air treatment member of claim 14, wherein the first dirt outlet region has a long dimension having a radial projection that is aligned parallel to the cyclone longitudinal axis.

16. The cyclonic air treatment member of claim 1, wherein the cyclone further comprises a third dirt outlet region to the dirt collection chamber, the third dirt outlet region is formed in the cyclone sidewall, and is oriented transverse to the first and second dirt outlet regions.

17. The cyclonic air treatment member of claim 16, wherein the first, second, and third dirt outlet regions are contiguous.

18. The cyclonic air treatment member of claim 16, wherein the first, second, and third dirt outlet regions are discrete.

19. The cyclonic air treatment member of claim 1, wherein the first dirt outlet comprises a plurality of apertures provided in the cyclone sidewall.

20. The cyclonic air treatment member of claim 1, wherein the cyclone first end is closed.

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21. A cyclonic air treatment member comprising:

- (a) a cyclone having a cyclone sidewall, a cyclone first end, an opposed cyclone second end, a cyclone air inlet proximate the cyclone first end, a cyclone air outlet proximate the cyclone second end and a cyclone longitudinal axis extending from the cyclone first end to the cyclone second end, wherein a cyclone chamber is located between the cyclone first and second ends and the cyclone sidewall has a plurality of dirt outlets;
- (b) a dirt collection chamber external to the cyclone chamber and in communication with the plurality of dirt outlets; and,
- (c) a suction motor downstream from the cyclone air outlet whereby air which has exited that cyclone air outlet continues downstream to the suction motor in the absence of reentering the cyclone chamber,

wherein the cyclone air outlet extends into the cyclone chamber from the cyclone second end and the plurality of dirt outlets are positioned radially outwardly from the cyclone air outlet and a plane that is transverse to the cyclone longitudinal axis extends through the cyclone air outlet and at least one of the plurality of dirt outlets.

22. The cyclonic air treatment member of claim 21, wherein the cyclone first end is closed.

23. A cyclonic air treatment member comprising:

- (a) a cyclone having a cyclone sidewall, a cyclone first end, an opposed cyclone second end, a cyclone longitudinal axis extending from the cyclone first end to the cyclone second end, a cyclone air inlet and a cyclone air outlet which extends into a cyclone chamber from the cyclone second end, the cyclone air outlet comprising a porous member, wherein the cyclone sidewall has a plurality of dirt outlets that are positioned radially outwardly from the porous member and a plane that is transverse to the cyclone longitudinal axis extends through the cyclone air outlet and at least one of the plurality of dirt outlets;
- (b) a dirt collection chamber external to the cyclone chamber and in communication with the plurality of dirt outlets; and,
- (c) a suction motor downstream from the cyclone air outlet whereby air which has exited that cyclone air outlet continues downstream to the suction motor in the absence of reentering the cyclone chamber.

24. The cyclonic air treatment member of claim 23, wherein the porous member comprises a screen or a shroud.

25. The cyclonic air treatment member of claim 23, wherein the cyclone first end is closed.

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